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A COMPUTER PROGRAM FOR FUEL SLOSHING IN AN AXISYMMETRIC TANK

by

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TECHNICAL REPORT NO. 10

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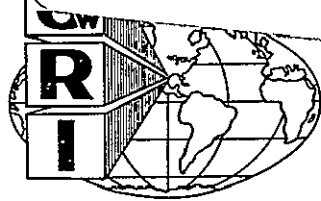
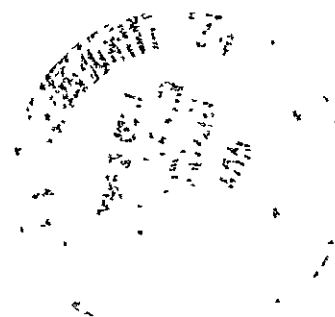
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I. DESCRIPTION OF MAIN PROGRAM

A. Program Name

The name of the main program is TRIPOT, meaning a potential problem solved by a non-uniform triangular mesh.

B. Functions of TRIPOT

1. Generation of a Non-Uniform Triangular Mesh

The non-uniform triangular mesh is generated on computer by the Winslow method (Ref. 1). The essence of the method is to use a transform to a logical plane, ξ, η which is related to the physical plane x, y by

$$\frac{\partial^2 \xi}{\partial x^2} + \frac{\partial^2 \xi}{\partial y^2} = 0, \quad \frac{\partial^2 \eta}{\partial x^2} + \frac{\partial^2 \eta}{\partial y^2} = 0 \quad (1a, b)$$

and by interchanging dependent and independent variables

$$\alpha x_{\eta\eta} + \beta x_{\xi\eta} + \gamma x_{\xi\xi} = 0 \quad (2a)$$

$$\alpha y_{\eta\eta} + \beta y_{\xi\eta} + \gamma y_{\xi\xi} = 0 \quad (2b)$$

where

$$\alpha = x_{\xi}^2 + y_{\xi}^2 \quad (3a)$$

$$\beta = x_{\eta}x_{\xi} + y_{\eta}y_{\xi} \quad (3b)$$

$$\gamma = x_{\eta}^2 + y_{\eta}^2 \quad (3c)$$

On the logical plane ξ, η ; the domain is composed of equilateral triangles only.

Equations (2a, b) are solved by specifying boundary values of x, y on the selected logical domain ξ, η . For fuel sloshing in an axisymmetric tank we chose a simple parallelogram as our logical domain (Fig. 3, Appendix II). * The interface of arc length s_1 is located on $i = 1, j = 1$ to N . The centerline of length s_2 is between the interface and the tank bottom. The wall is divided into two parts; the "bottom" of arc length s_3 and the "side" of arc length s_4 . For cylindrical tanks, the side and bottom are distinct; hence it will be used as such and not be redefined, and then s_2 and s_4 are divided into M parts, while s_3 is divided into N parts as s_1 . For a tank such as a spherical tank, the wall is arbitrarily divided into s_3 and s_4 approximately in the ratio s_1 to s_2 . Parts of approximately equal length can be specified. For fuel sloshing, especially with a "folding" interface, more accuracy near the contact point and the folding part may be desirable (as the machine time and the storage locations increase rapidly with number of points on the interface and the side wall). Then one may use a coarser net near the center on the interface and away from the contact point on the side wall, while s_2 and s_3 are divided in parallel with s_4 and s_1 , respectively. One may also use a finer net near the junction of s_3 and s_4 . In general, it is preferable to generate nearly equilateral triangles and to avoid obtuse triangles (not necessarily eliminated in the Winslow method). For a folding interface, relatively high accuracy of the interface points is needed, and can be fed in from outputs of the interface program, SSHAPE, meaning surface

*All figures in the theory are given in Appendix II.

shape (see Appendix I). For example, one may use $N = 17$ in SSHAPE (there are 16 intervals but take 5 double intervals near the center and 6 single intervals elsewhere), then in TRIPOT use $N = 12$ (there are 11 intervals). Likewise, a net of $N = 24$ may be obtained with piecewise uniform spacing.

With boundary values of x , y specified, the interior net points are first solved by a "linearized" approximation (Ref. 1), namely

$$\frac{\partial^2 x}{\partial \xi^2} + \frac{\partial^2 x}{\partial \eta^2} = 0, \quad \frac{\partial^2 y}{\partial \xi^2} + \frac{\partial^2 y}{\partial \eta^2} = 0 \quad (4a, b)$$

Then the technique of combined underrelaxation of coefficients and overrelaxation of dependent variables is used to solve Eq. (2a, b) starting with solutions of Eq. (4a, b). For Eq. (4a, b), an initial relaxation factor of $WA = 1.8$ is used. For Eqs. (2a, b), an initial overrelaxation factor of $WB = 1.0$ is used. The overrelaxation factors are improved as given in reference 1. For example,

$$\eta_{Winslow}^n = ETAX = \left[\frac{\sum_i \sum_j (x_{ij}^n + 1 - x_{ij}^n)^2}{\sum_i \sum_j (x_{ij}^n - x_{ij}^n - 1)^2} \right]^{1/2} \quad (5)$$

$$\lambda_{Winslow} = ELX = \left[\frac{\omega^n + \eta^n - 1}{\omega^n \sqrt{\eta^n}} \right]_{Winslow} = \frac{WX + ETAX - 1}{WX * \sqrt{ETAX}} \quad (6)^\dagger$$

[†]WX is the overrelaxation factor.

$$\begin{aligned}
(\omega'_{\text{opt}})_{\text{Winslow}} &= \text{WAX} = \left[\frac{2}{1 + (1 - \lambda^2)^{1/2}} - \omega_0 \right]_{\text{Winslow}} \\
&= \frac{2}{1 + \sqrt{1 - \text{ELX}^2}} - \text{W0}; \quad \text{W0} \cong 0.01 \quad (7)
\end{aligned}$$

$$\begin{aligned}
(\omega^{n+1})_{\text{Winslow}} &= \text{WX} = [\beta \omega'_{\text{opt}} + (1 - \beta) \omega^n]_{\text{Winslow}} \\
&= \text{RHO} * \text{WAX} + (1 - \text{RHO}) \text{WX}; \quad \text{RHO} = 0.05 \quad (8)
\end{aligned}$$

It is noted that if $\text{ETAX} > 1$, use $\text{WAX} = \text{WX}$ (previous) instead of Eq. (7).

Let

$$\begin{aligned}
\text{XTEMP} &= [C_1 x_{i-1,j}^n + C_2 x_{i-1,j-1}^n + C_3 x_{i,j-1}^n \\
&\quad + C_4 x_{i+1,j}^n + C_5 x_{i+1,j+1}^n + C_6 x_{i,j+1}^n] / \text{SUMC} \quad (9)
\end{aligned}$$

$$\text{SUMC} = \sum_{k=1}^6 C_k \quad (10)$$

$$\text{RX} = [x_{i,j}^n - \text{XTEMP}] * \text{WX} \quad (11)$$

then

$$x_{i,j}^{n+1} = x_{i,j}^n - \text{RX} \quad (12)$$

It is noted that in the present problem, we use a simple parallelogram in the logical plane (Fig. 3, Appendix II). Two examples of constructed nets in a physical plane are shown in Figures 4a and 4b, Appendix II.

2. Sequence of Calculations

The theory using auxiliary characteristic functions ψ_j is given in Appendix II. They satisfy $\frac{\partial \psi_j}{\partial n} = \lambda \psi_j$ on F , $\frac{\partial \psi_j}{\partial n} = 0$ on W and $\nabla^2 \psi = 0$ in V_L .

In our program, ψ_j is found by the influence coefficient method. The influence coefficients are F_{jk} and, therefore, one has

$$\psi_j = F_{jk} \frac{\partial \psi_k}{\partial n} = \lambda F_{jk} \psi_k \text{ on } F \quad (13)$$

or

$$([F] - \frac{1}{\lambda} [I]) \{\psi\} = 0 \text{ on } F \quad (14)$$

which is a standard eigenvalue problem; the k^{th} eigenvector being ψ_k , and the corresponding λ_k are printed as eigenvalues. The k^{th} column of F_{jk} is constructed with a unit k^{th} element of $\frac{\partial \psi}{\partial n}$ and zero normal derivative elsewhere. The value of k varies from 2 to N while $\psi(1, 1) = 0$ and $\frac{\partial \psi(1, 1)}{\partial n} = 0$, at the center of F . ψ is governed by

$$\nabla^2 \psi = 0 \quad (15)$$

A finite difference formula of Eq. (15) on the triangular mesh was given in reference 1. Let

A_{ij} = the area of ij^{th} dodecagon inside the fluid domain

r_{ij} = radius of the ij^{th} point

$w_k = \frac{1}{2} (\lambda_k \bar{r}_k \cot \theta_k + \lambda_{k-1} \bar{r}_{k-1} \cot \sigma_k)$, $k = 1$ to 6

(see Fig. 1a, b)[†]

$\bar{r}_k = \frac{1}{3} (r_{1j} + r_k + r_{k+1})$

[†] θ_k, σ_k can be expressed in terms of $t_k, u_k, t_{k-1}, u_{k-1}$ and s_k .

For an interior point (i, j),

$$\sum_{k=1}^6 w_k (\psi_k - \psi) - \frac{m^2}{r_{ij}} A_{ij} \psi = 0, \quad m = 1 \quad (16)$$

with

$$\lambda_k = 1$$

For an interface point (1, j),

$$\sum_{k=1}^6 w_k (\psi_k - \psi) - \frac{m^2}{r_{1j}} A_{1j} \psi + \left(\frac{\partial \psi}{\partial n} \right)_{1j} \left[\frac{1}{2} s_3 + \frac{1}{2} s_6 \right] r_{1j} = 0 \quad (17)$$

with

$$\lambda_6 = \lambda_1 = \lambda_2 = 0, \quad \lambda_3 = \lambda_4 = \lambda_5 = 1, \quad i = 1$$

For ij^{th} point on the tank wall,

$$\sum_{k=1}^6 w_k (\psi_k - \psi) - \frac{m^2}{r_{ij}} A_{ij} \psi = 0, \quad m = 1 \quad (18)$$

with

$$\lambda_3 = \lambda_4 = \lambda_5 = 0 \text{ and } \lambda_1 = \lambda_2 = \lambda_6 = 1 \text{ on bottom wall,}$$

$$\text{i. e., } i = M, j = 1 \text{ to } N$$

$$\lambda_4 = \lambda_5 = \lambda_6 = 0 \text{ and } \lambda_1 = \lambda_2 = \lambda_3 = 1 \text{ on side wall, i. e.,}$$

$$j = N, i = 1 \text{ to } M$$

$$\lambda_1 = \lambda_2 = 1 \text{ and } \lambda_3 = \lambda_4 = \lambda_5 = \lambda_6 = 0 \text{ at corner point,}$$

$$i = M, j = N$$

On centerline,

$$\psi = 0 \text{ for } m = 1 \quad (19)$$

At contact point, $i = 1, j = N$

$$\sum_{k=1}^6 w_k (\psi_k - \psi) - \frac{m^2}{r_{ij}} A_{ij} \psi + \left(\frac{\partial \psi}{\partial n} \right)_{1,N} \left(\frac{1}{2} s_3 \right) r_{ij} = 0 \quad (20)$$

with

$$\lambda_3 = 1 \text{ and } \lambda_4 = \lambda_1 = \lambda_2 = \lambda_5 = \lambda_6 = 0$$

Equations (16) to (20) are solved by the overrelaxation method through use of subroutine SOREL (1), which contains the same procedure and factors as those used in solving for x [see Eqs. (8), (12)].

After the influence coefficient matrix is determined, Eq. (14) determines the eigenvalues and eigenvectors (the auxiliary characteristic functions ψ_j on the interface). The values of ψ_j on the wall are obtained by the S.O.R. method through subroutine SOREL (2).

Then subroutine SOREL (1) is again used to solve the boundary value problem of a capped fluid (rigid interface) under pitching oscillation and subsequently the corresponding moment of inertia.

Next, the normal modes are determined by an expansion of auxiliary characteristic functions (using N_{ev} numbers[†] out of the N number

[†]It is probably more accurate not to use the last few auxiliary eigenfunctions which contain larger errors.

of ψ_j^i s). A matrix eigenvalue problem results as given in Appendix II. The computer program thus calculates various integrals entering the matrix equation. An eigenvalue subroutine yields the coefficients in series expansion of the normal modes (eigenvectors) and the corresponding frequency (eigenvalue).

Once the coefficients in the series expansion of the normal modes are determined, integrals containing these functions (Appendix II) can be evaluated to determine the parameters of the spring-mass mechanical model. This completes the main steps of the present program.

II. EXAMPLES

Results of numerical examples are given in Appendix III.

It is noted that for a spherical tank with a "folding" interface, the result is sensitive to the inputs, especially the interface points; piecewise uniform spacings of the interface points were used. With a 12×12 mesh, it takes about two minutes CDC 6600 time in the main program, TRIPOT. With 23×34 mesh, it would take about twenty minutes.

III. CONCLUSIONS

The program seems to yield reasonably good results in the examples calculated with as few as 12 interface points. However, in some cases, it is sensitive to inputs which should be reasonably accurate. Applications to other cases remain to be examined.

IV. INSTRUCTIONS FOR USING COMPUTER PROGRAMS "TRIPOT" AND "SSHAPE"

A. General

TRIPOT is a program that computes the sloshing parameters for an axisymmetric tank. A finite-difference technique using a triangular mesh is used to solve the potential flow equations. The main restriction on the program is that the Bond number and the volume of contained liquid are both great enough so that the liquid covers the bottom of the tank.

SSHAPE is a program used to compute the equilibrium free surface shape of the liquid. The output of SSHAPE is used as input for TRIPOT.

B. TRIPOT

A triangular mesh is laid over the liquid by a method to be described. Figures 4a and 4b, in Appendix III, show typical meshes for a cylindrical and a spheroidal tank.

Both the liquid interface and the tank shape are given as discrete x, y coordinates, $x = 0, y = 0$ being the point where the free surface intersects the tank centerline. The interface, arc length s_1 , is broken up into N points. The centerline, arc length s_2 , is broken up into M points. The bottom, s_3 , and side wall, s_4 , are specified at N and M points, respectively. For a spheroid, for example, in which there is no "bottom" or "side," the wall is arbitrarily divided into arc lengths s_3 and s_4 such that s_3/s_4 is approximately equal to s_1/s_2 .

For a "folded over" interface, additional accuracy can be obtained by using a non-uniform spacing of the N and M points. A coarse net is used on the interface, s_1 , near the centerline and on the part of the wall not close

to the contact point, s_4 . Then, s_2 and s_3 are divided similarly to s_1 and s_4 , respectively. For example, 16 intervals ($N = 17$) can be used in SSHAPE to calculate the free surface shape. Then, the s , y coordinates of the 1, 3, 5, 7, 9, 11, 12, 13, 14, 15, 16, 17 points can be used as input to TRIPOT with $N = 12$ (eleven intervals, 5 wide ones near the centerline and 6 narrow ones near the contact point).

The interior grid points are generated by TRIPOT. In general, it is better to use near-equilateral triangles and avoid obtuse ones. Some trial-and-error work may be necessary to insure this.

Subprograms Used

SOREL -	Solution of a system of equations by successive over-relaxation
SFIT -	Curve fitting by a second degree polynomial
EIGEN -	Eigenvalues and vectors of a real symmetric or non-symmetric matrix
MATINV -	Matrix inversion routine

Tapes Used

TAPE 6 - Restart write tape
 TAPE 7 - Restart read tape

Storage Used

Less than 32,000 case storage locations

Input Formats Used

<u>Card No.</u>	<u>Format</u>
1	(12A6)
2	(4I5, 5F10.0)
3†	(8F10.0)
4†	(8F10.0)
5†	(8F10.0)
6†	(8F10.0)
7	(8F10.0)
8	(5E15.8)
9	(4I5, 5F10.0)

C. TRIPOT--Program Restart Procedure

At seven different places in the program, results calculated thus far are saved on tape so that it is not necessary to calculate from the beginning in the event of catastrophic failure or insufficient allotted time. When using the restart feature it is necessary to interchange tape six with tape seven so that the tape which was written will become the tape being read. One method of effecting the interchange is by redefining LUN and KUN, which are the first two executable statements in the main program. During execution, the printed output will state the highest restart number which can be used. The input data card numbers to be used with each restart are given below.

<u>Restart Number</u>	<u>Input Data Card Numbers</u>
1	1, 2, 7, 8, 9
2	1, 2, 7, 8, 9
3	1, 2, 7, 8, 9
4	1, 2, 8, 9
5	1, 2, 8, 9
6	1, 2, 8, 9
7	1, 2, 8, 9

†Set of cards.

TABLE I. INPUT OF TRIPOT

TRIPOT --Input Data Description

<u>Card No.</u>	<u>FORTRAN Symbol</u>	<u>Variable Name</u>	<u>Units</u>	<u>Definition</u>
1	ITITLE			72 columns of identification
2	M			Number of centerline points
	N			Number of interface points
	ITA			Maximum number of iterations
	IRS			Restart number (initially zero)
	WA		non-dim.	Relaxation factor for linear approximations
	WB		non-dim.	Initial relaxation factor for nonlinear solution
	EPS		non-dim.	Convergence factor
	RLGTH		length	Reference length used for non-dimensionalization, e. g., maximum tank radius
	BOND	N_B	non-dim.	Bond number
3†	X(1, J), Y(1, J)	(x, y) _{j=1, N}	length	Coordinates for interface‡; j = 1 at tank center, increasing outward
4†	X(I, N), Y(I, N)	(x, y) _{i=2, M-1}	length	Coordinates for wall; i = 1 at interface increasing downward
5†	X(M, J), Y(M, J)	(x, y) _{j=1, N}	length	Coordinates for centerline, j = 1 at tank center, increasing outward
6†	X(I, 1), Y(I, 1)	(x, y) _{i=2, M-1}	length	Coordinates for centerline, i = 1 at interface increasing downward

†Set of cards.

‡It is recommended to use 5 digits (or more) obtained from output of interface program. Wall coordinates are not as critical.

TABLE I (cont'd)

<u>Card No.</u>	<u>FORTTRAN Symbol</u>	<u>Variable Name</u>	<u>Units</u>	<u>Definition</u>
7	GFLC		non-dim.	= 0, compute Γ ; input dummy Γ = 1, input Γ from SSHAPE output, or if interface is flat input $\Gamma = 0$.
	GAMMA	Γ	non-dim.	Contact point constant
	THETAC	θ_c	degrees	Contact angle; must be greater than 2°
8	ZCG		length	Center of gravity of liquid†
	VOL		length ³ cube	Volume of liquid†
	DEN		mass/length ³ cube	Density of liquid
	RHOU		mass/length ³ cube	Density of ullage fluid
9	NEV	N_{ev}	non-dim.	Number of eigenvectors used

†Obtained as output from SSHAPE.

D. TRIPOT Input Example and General Instructions

Example for a Spheroidal Tank and General Input† Instructions are given below:

1. TEST CASE xx, x Full, Bond number = x (see input description)
2.
 - a. M = 9 no. of side wall points, see 5 below
 - b. N = 12 no. of interface points selected, see 4 below; usually greater than 10
 - c. ITA = 200 suggested value; usually sufficient
 - d. IRS = 0 always zero unless using restart procedure
 - e. WA = 1.8 suggested value
 - f. WB = 1.3 suggested value
 - g. EPS = 1×10^{-6} suggested value
 - h. RLGTH = 1.0 (see input data description)
 - i. Bond = 5 specified value
3. X(1, J), Y(1, J) selected interface points from output of SSHAPE
4. X(I, N), Y(I, N) side wall points selected after interface is calculated; a plot of interface, tank wall and center is desirable for selecting wall points. Suggest division near contact point be nearly an isocircle or form an acute triangular. It is not necessary to have precisely equal arc divisions. Usually, the meshes may be finer near the contact point. The points can be read off from curve (or calculated from analytic expression); accuracy is not as critical as the interface points.

†TRIPOT input data descriptions are given in Table I.

5. $X(M, J), Y(M, J)$ bottom wall points; the ratio of bottom wall to side wall in arc length is approximately that of interface to wetted centerline (eye observation is usually sufficient). The divisions on the bottom are suggested to be approximately parallel to that on the interface. Points can usually be read off from curve plotted.

6. $X(I, 1), Y(I, 1)$ wetted centerline points. The divisions on the centerline is suggested to be approximately parallel to that on the side wall.

7. a. $GFLC = 1.0$ (see input data description) unity using output from SSHAPE

- b. $GAMMA = -16.12$ output CGAM from SSHAPE

- c. $THETAC = 5^\circ$ specified contact angle (no less than 2°)

8. a. $ZCG = -.01$ output from SSHAPE; c.g. of liquid, negative below center of interface

- b. $VOL = 1.35$ output from SSHAPE; liquid volume

- c. $DEN = 1.0$ if $RHOU = 0$, the density of the liquid DEN can be set to 1 without affecting result

- d. $RHOU = 0$ density of ullage fluid usually neglected (zero)

9. $NEV = 7$ number of eigenvectors used; for $N \geq 11$, $N_{ev} = 7$ has been selected. For partial convergence test on a finer mesh $N_{ev} = 7$ can be retained and for the convergence test on Galerkin procedure, N_{ev} may be increased. However, it may be advisable to use $N_{ev} < N-3$ or smaller to avoid large inaccuracy in the higher auxiliary characteristic functions.

E. TRIPOT Output

TRIPOT--Program Printed Output

1. Input Data - ITITLE, M, N, ITA, WA, WB, EPS, IRS

2. (x, y) coordinates for linear approximations to the mesh

3. (x, y) coordinates for nonlinear solution to the mesh
4. Input Data (compute option) - GAMMA
5. Eigenvalues λ_j and eigenvectors ψ_{mj} on F
6. Input Data - BOND, ZCG, VOL, DEN, RHOU
7. Eigenvalues $\Omega_k^2 = \frac{\omega_k^2}{g}$ and eigenvectors C_{kj}
8. SIF, $I_F/M_F h_o^2$
9. SIO, $I_o/M_F h_o^2$
10. MK/MF_1 - nondimensional i^{th} slosh mass, M_K/M_F
11. ZK_i , k^{th} slosh mass location, z_k with $i = k = 1, 2 \dots NEV$
12. Z0, location of rigid mass, z_0

REFERENCES OF THE MAIN COMPUTER PROGRAM

1. Winslow, A. M., "Numerical Solutions of the Quasi-Linear Poisson Equations in a Non-Uniform Triangular Mesh, " Journal of Computational Physics, Vol 1, No. 2, pp 149-172, November 1966.
2. See references in Appendix II.

APPENDIX I

A COMPUTER PROGRAM FOR GENERATING THE INTERFACE SHAPE AND ASSOCIATED QUANTITIES

A. Governing (Mean) Interface Equation

The interface differential equation is given by†

$$y'' = \frac{2y^2 + 3(y')^2}{y} - \frac{y'}{y^2} \cot(\theta) [y^2 + (y')^2] + \frac{1}{y} \left[B_n(y \cos(\theta) - y_0) - \frac{2k_0}{y_0} \right] [y^2 + (y')^2]^{3/2} \quad (I-1)$$

Converting from second order to first order,

let

$$\begin{aligned} y_1 &= y \\ y_2 &= y' \end{aligned}$$

then

$$\begin{aligned} y_1' &= y' \\ y_2' &= y'' \end{aligned} \quad (I-2)$$

or

$$\begin{aligned} y_1' &= y_2 \\ y_2' &= \frac{2y_1^2 + 3y_2^2}{y_1} - \frac{y_2}{y_1^2} \cot(\theta) [y_1^2 + y_2^2] + \frac{1}{y_1} \left\{ B_n[y_1 \cos(\theta) - y_0] - \frac{2k_0}{y_0} \right\} (y_1^2 + y_2^2)^{3/2} \end{aligned} \quad (I-3)$$

†Hastings, L. J. and Rutherford, R., III, "Low Gravity Liquid-Vapor Interface Shapes in Axisymmetric Containers and a Computer Solution," NASA Technical Memorandum, NASA TM X-53790, October 7, 1968. In the equation to follow, y is the radial distance from the center of the top of the tank, θ is the angle counterclockwise from the tank centerline with origin at the center of the top.

The solution of the first order differential equations (I-2) and (I-3) is obtained by using the Runge-Kutta-Gill fourth order method; the boundary of the tank is approximated by a polygon. In general, a new polygonal approximation is required below the interface according to the description aforementioned in the main text.

B. Program Notes

In addition to the main program, the following subprograms were used:

RKLDEQ - Runge-Kutta-Gill differential equation solver

SFIT - curve fitting by a 2nd degree polynomial.

The input data description of program SSHAPE is given in Table II and the output of program SSHAPE is given in Table III, respectively. Some of the program symbols which were not given previously are defined in Table IV and an example keypunch form is given in Table V.

C. Input Instructions for SSHAPE

The input for a spherical tank and general instructions are given below:

- | | |
|--------------------------|---|
| 1. NP = 37 | Total number of points specifying the tank starting from center of bottom to center of top; varies with each case |
| 2. YBX _i | Horizontal and vertical coordinates of tank; should be a close polygonal approximation to the tank, i = 1 starting from center of tank bottom |
| 3. YBZ _i | |
| 4. a. $\alpha = 5^\circ$ | Contact angle specified |
| b. KO = 0 | Usually zero |

- c. $YO = 1.0$ Initial guess at distance between center of interface and center of tank top
- d. $BN = 5$ Bond number specified
- e. $BETAD = 5/8 = .625$ Empty fraction specified
- f. $DKO = 0.2$ Suggested
- g. $DBC = .005$ Convergent criterion
- h. $DYO = .05$ Suggested
- 5. a. $THETA = 0$ Usually zero
- b. $DTHETA = .05$ Suggested
- c. $RLGTH = 1$ (See input data description)
- 6. a. $N = 2$ Always
- b. $IBOPT = 1$ Unity if calculating empty fraction (see input data description)
- 7. a. $NN(1) = 17$ (Only 12 of these 17 were selected for input in TRIPOT)
- b. $NN(2) = 33$ For a finer mesh if desired

D. Listing and Sample Input

These are included after those of TRIPOT at the end of the report.

TABLE II

INPUT DATA DESCRIPTION OF PROGRAM SSHAPE

<u>Card No.</u>	<u>FORTTRAN Symbol</u>	<u>Variable Name</u>	<u>Units</u>	<u>Definition</u>
1	NP			No. of points on container boundary
2	YBX ₁		length	x-coordinate container boundary, positive outward from top center, i = 1 at bottom center
3	YBZ ₁		length	z-coordinate container boundary, positive downward from top center, i = 1 at bottom center
4	ALPHA	α	degrees	Desired contact angle, θ_c , must be greater than 2°
	KO	K_o	non-dim.	Assumed initial guess of parameter related to curvature k_o at interface center point, i.e. $k_o y_o$. Usually zero can be used.
	YO	y_o	length	Estimated distance from center of tank top to interface centerpoint
	BN	B_n	non-dim.	Bond number, N_{Be}
	BETAD	β_D	non-dim.	Desired empty fraction
	DKO	ΔK_o	non-dim.	Increment for K_o , a typical value is 0.2.
	DBC	$\epsilon_{\beta D}$	non-dim.	Convergence criterion for β_D . (A typical value is 0.005)
	DYO	Δy_o	non-dim.	Increment for y_o . (A typical value is $0.05 R_o$).
5	THETA	θ	degrees	Initial angle measured from vertical axis to y . Usually zero.

TABLE II
INPUT DATA DESCRIPTION OF PROGRAM SSHAPE (Cont'd)

<u>Card No.</u>	<u>FORTRAN Symbol</u>	<u>Variable Name</u>	<u>Units</u>	<u>Definition</u>
	DTHETA	$\Delta\theta$	degrees	Increment for θ . (A typical value is .05)
	RLGTH	R_o	length	Characteristic container dimension
6	N	n		No. of equations
	IBOPT			Option for calculation of empty fraction β . If = 0, suppress calculation [†] ; if = 1, calculate β
7	NN(1)			Two choices of no. of interface points of equal intervals of which the locations are desired.
	NN(2)			

[†]In case y_o is known from either another theory or experiments, one may proceed the calculation with a dummy mass fraction. In theoretical predictions, usually desired empty fraction is given, then we must use IBOPT = 1 to calculate β for each assumed y_o in the program, while the input y_o is an approximate guess.

TABLE III. OUTPUT OF PROGRAM SSHAPE

Printed Output

1. Input data ALPHA, KO, YO, BN, THETA, Y(1), Y(2)
2. THETA, XOR, ZOR, Y(1), Y(2)
3. GAMMA, PHI, ALPHA, YO, KO
4. Extrapolated YO (non-dimensional)
5. VU1, VU2, VU, VL, VT
6. BETA
7. G, RS, S, XR, XY
8. K2F, FRR2 (interface curvature n_F and $\frac{d^2 F}{dR^2}$ on F at the contact point)
9. K2B, ZRR2 (wall curvature n_B and $\frac{d^2 Z}{dR^2}$ on wall at the contact point)
10. CGAM (Γ)
11. Dimensional YO
12. XRB, XYB
13. ZCGT, BZCGU, ZCG

<u>FORTTRAN Symbol</u>	<u>Variable Name</u>	<u>Units</u>	<u>Description</u>
S_1	S	length	Arc length, $i = 1$ at interface center point
G_1	G	length square	Sum of curvatures squared, $i = 1$ at interface center point
RS_1	R_s	length	dR/dS , $i = 1$ at interface center point

TABLE III. OUTPUT OF PROGRAM SSHAPE (Cont'd)

<u>FORTTRAN</u> <u>Symbol</u>	<u>Variable</u> <u>Name</u>	<u>Units</u>	<u>Description</u>
CGAM	Γ	non-dim.	Contact point constant
XR	XR	length	Horizontal distance from container vertical axis to liquid-vapor interface with origin at interface center point
XY	XY	length	Vertical distance from x-axis to liquid vapor interface with origin at interface center point
XRB	XRB	length	x-coordinates of container boundary with origin at interface center point
XYB	XYB	length	y-coordinate of container boundary with origin at interface center point
ZCGT	Z_{CGT}	length	Container c. g.
BZCGU	βZ_{CGU}	length	Mass fraction times distance of ullage c. g. upward from interface center point
ZCG	Z_{CG}	length	Distance of liquid c. g. upward positive from interface center point

TABLE IV. DEFINITION OF TERMS IN SS SHAPE

Some of the program symbols which were not defined previously.

<u>FORTTRAN Symbol</u>	<u>Variable Name</u>	<u>Units</u>	<u>Definition</u>
XOR	X/R_o	non-dim.	Horizontal distance from container vertical axis to interface point
ZOR	Z/R_o	non-dim.	Vertical distance (downward positive) from x-axis to interface point, origin at top center.
Y(1)	y	non-dim.	Radial distance from top center to interface point
Y(2)	y'	non-dim.	First derivative of y with respect to θ
GAMMA	γ	degrees	Angle measured from vertical and tangent to $y(\theta)$ at contact point
PHI	ϕ	degrees	Angle measured from vertical and tangent to $y_B(\theta)$ at contact point
VU1		length cube	Vapor volume to point of contact
VU2		length cube	Vapor volume from point of contact to boundary
VU		length cube	Vapor volume
VL		length cube	Liquid volume
VT		length cube	Total container volume

TABLE V. EXAMPLE KEYPUNCH FORM FOR SSHAPE

C COMMENT		CONTINUATION		COMPUTATIONS LABORATORY		PROBLEM Example Keypunch Form		PAGE OF						
				SOUTHWEST RESEARCH INSTITUTE		PROGRAMMER		DATE 8/20/69						
				FORTRAN		STATEMENT								
1		5	6	7	10	20	30	40	50	60	70	72	73	80
	NP	1			FORMAT (2I5)									
	3													
	YBX(I), I=1, NP				FORMAT (8F10.0)									
0.					0.68	0.68								
	YBZ(I), I=1, NP				FORMAT (8F10.0)									
2.04					2.04	0.								
ALPHA, KIO, YO, BN, BETAD, DKO, DBC, DYO					FORMAT (8F10.0)									
2.					0.	0.45	100.	0.4	0.2	0.005	0.05			
THETA,					DTHEA,	RLGTH	FORMAT (8F10.0)							
0.					0.05	0.68								
	N,				IBOPT	FORMAT (2I5)								
	2				0									
NN(1),					NN(2)	FORMAT (2I5)								
23					0									

APPENDIX II

A THEORY[†] FOR LOW-GRAVITY FUEL SLOSHING IN AN
ARBITRARY AXISYMMETRIC RIGID TANK

Wen-Hwa Chu

[†]This theory and main results have already been published as ASME paper 70-APM-EEE to appear in the Journal of Applied Mechanics and contains major extensions and modifications of Technical Report No. 8. It is included for convenience of the sponsor and the readers of this report.

Introduction

THE behavior and consequences of fuel sloshing in rockets under a high effective gravity were recognized problems which have been quite well understood [1-3].¹ The problem of low-gravity fuel sloshing, characterized by the significant role of interfacial tension, is now a subject of importance for application to coasting rockets or orbital stations.

The equilibrium behavior of fluids at zero and/or low gravity has been studied in references [4-7]. The theoretical determination of an equilibrium interface shape is nonlinear and requires a trial and error procedure for a given contact angle [5, 6].

Satterlee and Reynolds [8] have successfully solved the free sloshing problem in cylindrical containers under low gravity and formulated a variational principle for this purpose. Yeh [9], using a similar approach, solved the free and forced sloshing problem under low-gravity conditions, without force and moment or an equivalent mechanical model. Dodge and Garza [10, 11] performed force measurements under simulated low-gravity conditions and predicted forces and moment for circular cylindrical tanks under lateral (translational) motion. The equivalent spring-mass model was given in [10]. Additional work by Dodge and Garza for other special tanks was given in [12, 13]. A finite-difference approach with application to a hemispherically bottomed cylindrical tank and spheroidal tanks was given by Concus, Crane, and Satterlee in [14, 15].

These investigations indicate a need of a program for a general axisymmetric tank. A preliminary study on liquid sloshing in an arbitrary axisymmetric tank was reported in [16], but it is

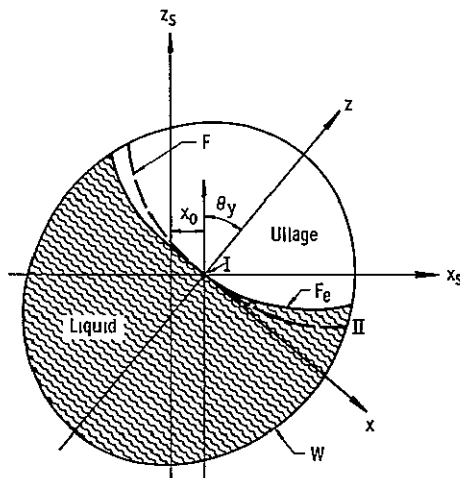


Fig. 1 Some Nomenclatures

limited to translational oscillations. It is the object of the present paper to present a seminumeral approach for an arbitrary axisymmetric tank with simplified force and moment calculations and the resultant mechanical model for both pitching and translational oscillations. A general computer program utilizing Winslow method [17] will be completed to obtain sloshing frequencies, slosh mass, and mass-height, for which a brief description is given in Appendix A.

¹ Numbers in brackets designate References at end of paper. Contributed by the Applied Mechanics Division for publication (without presentation) in the JOURNAL OF APPLIED MECHANICS.

Governing Equations

Assuming irrotational incompressible flow, there is a space-fixed velocity potential² ϕ satisfying the Laplace equation in both space-fixed and tank-fixed coordinates.

$$\nabla_s^2 \phi = 0, \quad \nabla^2 \phi = 0,$$

$$\nabla_s^2 = \frac{\partial^2}{\partial x_s^2} + \frac{\partial^2}{\partial y_s^2} + \frac{\partial^2}{\partial z_s^2}, \quad \nabla^2 = \frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2} \quad (1)$$

As in thin airfoil theory, the velocity potential can be obtained by imposing boundary conditions on the initial or mean position, but the hydrostatic pressure due to gravity possesses components along both the tank axis z and the lateral axis x , Fig. 1, for pitching oscillations. The linearized Bernoulli's equation states

$$p - p_1 + \rho \frac{\partial \phi}{\partial t} + \rho g(z - x\theta_y) = 0 \quad (2)^3$$

and

$$p - p_{u1} + \rho_u \frac{\partial \phi_u}{\partial t} + \rho_u g(z - x\theta_y) = 0 \quad (3)$$

for the liquid and the ullage, respectively, and p_1, p_{u1} are constants.

Boundary Conditions

The linearized interface kinematic condition states that

$$\frac{\partial h}{\partial t} \approx \frac{\partial \phi}{\partial n} \quad \text{on } F \quad (4)^4$$

The interface dynamic condition states that

$$-p_- + p_+ = \sigma \kappa = \sigma \kappa_0 + \sigma \kappa' \quad \text{on } F_e \quad (5)$$

For the "mean" interface location f (in general, $p_1 = p_1^0 + p_1'$, p_1^0, p_1' being constants),

$$-\sigma \kappa_0 + (\rho - \rho_u)gf - (p_1^0 - p_{u1}^0) = 0 \quad \text{on } F \quad (6)$$

where the curvature of the mean interface, κ_0 , is axisymmetric and

$$\kappa_0 = \frac{1}{r} \frac{\partial}{\partial r} \left\{ -\frac{r \frac{\partial f}{\partial r}}{\sqrt{1 + \left(\frac{\partial f}{\partial r}\right)^2}} \right\} = \frac{1}{r} \frac{\partial f}{\partial s} + \left(\frac{\partial r}{\partial s} \frac{\partial^2 f}{\partial s^2} - \frac{\partial f}{\partial s} \frac{\partial^2 r}{\partial s^2} \right) \quad (6a)$$

Equation (6) holds for $r = 0$, thus

$$p_1^0 - p_{u1}^0 = -2 \left(\frac{\partial^2 f}{\partial r^2} \right)_{11}$$

The linearized interface dynamic condition is then

$$-(p_u' - p_1') - \sigma \kappa' + \rho \frac{\partial \phi}{\partial t} - \rho_u \frac{\partial \phi_u}{\partial t} + (\rho - \rho_u)gh \frac{\partial f}{\partial s} - (\rho - \rho_u)gx\theta_y = 0 \quad (7)^5$$

² For example, $\partial \phi / \partial x$ gives velocity component in x -direction with respect to space-fixed coordinates.

³ Chu, W. H., "Free Surface Condition for Sloshing Resulting From Pitching and Some Corrections," *ARS Journal*, Vol. 30, Nov. 1960, pp. 1093-1094.

⁴ A vertical interface description was used but is only successful when $(dF/dR)^{-1}$ does not vanish on the interface.

⁵ For sinusoidal oscillations and $m = 1, h = 0, \phi = \phi_u = 0, x = 0$, and $\kappa' = 0$ at point I, thus $p_1' - p_{u1}' = 0$. For other m values, $-p_1' + p_{u1}' = \sigma \kappa'$, which will be omitted until needed.

where the perturbation curvature for $\cos(m\theta)$ variation is to the first order

$$\kappa' = \frac{\partial^2 h}{\partial s^2} + \frac{1}{r} \frac{\partial r}{\partial s} \frac{\partial h}{\partial s} + \frac{1}{r^2} \frac{\partial^2 h}{\partial \theta^2} + \left[\left(\frac{1}{r} \frac{\partial f}{\partial s} \right)^2 + \left(\frac{\partial r}{\partial s} \frac{\partial^2 f}{\partial s^2} - \frac{\partial f}{\partial s} \frac{\partial^2 r}{\partial s^2} \right)^2 \right] h \quad (7a)^6$$

m being unity for lateral excitation of a rigid tank. At point I, the origin, $h = 0$, $\phi = 0$, $\kappa' = 0$, and thus $p_I = p_{uI}$. For most analyses, $p_u = 0$ was assumed. We shall assume the impulsive pressure in the ullage is negligible, i.e., $\phi_u = 0$. Then, for sinusoidal oscillations with h , ϕ proportional to $\cos(\omega t)$ and $\sin(\omega t)$, respectively, equations (7), (7a), and (4) yield

$$- \left\{ \frac{1}{R} \frac{\partial}{\partial s} \left(R \frac{\partial H}{\partial s} \right) - \frac{m^2}{R^2} H + GH \right\} + N_{Be} H \frac{\partial R}{\partial s} + \theta_y N_{Be} R \cos \theta + \Omega^2 \Phi = 0 \quad \text{on } F \quad (7b)^7$$

where

$$G = \left(\frac{1}{R} \frac{\partial F}{\partial s} \right)^2 + \left(\frac{\partial R}{\partial s} \frac{\partial^2 F}{\partial s^2} - \frac{\partial F}{\partial s} \frac{\partial^2 R}{\partial s^2} \right)^2 = \frac{1}{R^2} \frac{F_R^2}{(1 + F_R^2)} + \left[\frac{F_{RR}}{(1 + F_R^2)^{3/2}} \right]^2 \quad (7c)$$

It is noted that, for lateral oscillations, $m = 1$, solutions are proportional to $\cos \theta$.

The boundary condition on the wall is that the relative normal velocity be zero, i.e., with $\cos(n, x) = \frac{\partial x}{\partial n}$ and $\cos(n, z) = \frac{\partial z}{\partial n}$,

$$\frac{\partial \phi}{\partial n} = x_0 \frac{\partial x}{\partial n} \quad (8)$$

and

$$\frac{\partial \phi}{\partial n} = \theta_y \left(z \frac{\partial x}{\partial n} - x \frac{\partial z}{\partial n} \right) \quad (9)$$

for translational and pitching oscillations, respectively.

In addition, there is an interface contact point condition which takes the form [8, 9, and 15]

⁶ The instantaneous interface can be described by the position vector $r/a = [R - HF_s] \cos \theta \hat{i} + [R - HF_s] \sin \theta \hat{j} + [F + HR_s] \hat{k}$, $R_s = dR/ds$, $F_s = dF/ds$. The curvatures can then be derived from the fundamental magnitudes [18] neglecting higher-order terms. The linearized total curvature is given by this equation, which was first derived in [15] by a different procedure.

⁷ For sinusoidal oscillations, without loss of generality, x_0 , θ_y , ϕ are assumed to be proportional to $\sin(\omega t)$ while h is proportional to $\cos(\omega t)$.

Nomenclature

a = reference length, say, maximum radius of tank	p_{uI} = equilibrium ullage pressure at origin—a constant	Φ_L = amplitude of nondimensional potential $\phi_L/\omega a^2$
$d\Omega = r dr d\theta$	$R = r/a$, nondimensional radius	Φ_N = see equation (15)
$dA = d\Omega/a^2$ (a scalar)	r, θ, z = tank fixed cylindrical coordinates	Φ^0 = amplitude of nondimensional potential $\phi^0/\omega a^2$
$dS = 3-D$ surface element, e.g., $rd\theta ds$	s = arc length nondimensionalized by a	ϕ = velocity potential
$dS = dS/a^2$, nondimensional surface element (a scalar)	s = arc length (a scalar)	ϕ_L = velocity potential of the L th natural mode
F = equilibrium (mean) interface of f/a	t = time	ϕ' = additional velocity potential due to interface movement
F_e = instantaneous interface	V = volume of liquid divided by a^3	ϕ^0 = velocity potential of liquid with a frozen interface
F_H = horizontal force defined by equation (19)	V_I = liquid volume	ψ = velocity potential of auxiliary eigenfunctions
$F_R = (dF/dR)$, slope of F in the generatrix plane	W = wall wetted by liquid	ψ_j = j th auxiliary characteristic function
F_x = x -component of force on tank	W_e = instantaneous wetted wall below instantaneous interface, F_e	$\Omega^2 = \rho a^2 \omega^2 / \sigma$, product of Bond number, $\rho a^2 g / \sigma$, and frequency parameter, $\omega^2 a / g$
f = equilibrium (mean) interface elevation measured along vertical axis	x_0 = translational amplitude in x -direction	ω = frequency of oscillation
g = gravitational acceleration	x, y, z = space-fixed rectangular coordinates	ω_L = L th natural frequency
H = amplitude of h/a , nondimensional slosh height	Γ = γa , nondimensional contact point constant	
h = interface perturbation normal to equilibrium interface	γ = hysteresis coefficient or contact point constant	
h_0 = reference length, say, depth of liquid at center of tank	$\Delta\rho$ = density difference, $\rho - \rho_u$	
M_0, I_0 = rigid mass and moment of inertia of mechanical model	δ_{ij} = Kronecker delta	
M_L = liquid mass	ϵ_i = sign of $n \hat{z}$, $\cos(n, z)$, or $\partial z / \partial n$	
M_y = pitching moment about y -axis	θ_y = amplitude of pitching about y -axis	
m_L = L th slosh mass	λ = mean curvature	
N_B = Bond number, $\rho a^2 g / \sigma$	λ' = perturbation of mean curvature	
n = outer normal	λ_j = j th eigenvalue ($m = 1$)	
$n_0 = n/a$, nondimensional normal distance	λ_{mj} = j th eigenvalue corresponds to m th circumferential mode	
p = pressure	ρ = liquid density	
p_I = equilibrium liquid pressure at origin—a constant	ρ_u = density of ullage fluid (vapor or gas)	
p_u = ullage pressure	σ = surface tension	
	Φ = amplitude of nondimensional velocity potential, $\phi/\omega a^2$	

Subscripts

$()_I$ = $()$ at vertex of equilibrium interface (origin)
$()_{II}$ = $()$ at contact point in generatrix plane
$()_{GC}$ = $()$ related to center of gravity
$()_e$ = effective value of $()$
$()_F$ = $()$ on F
$()_m$ = $()$ associated with $\cos(m\theta)$ mode
$()_p$ = $()$ related to pitching
$()_T$ = $()$ related to translation
$()_n$ = $()$ on W
$()_u$ = $()$ related to ullage
$()_-$ = $()$ just below interface
$()_+$ = $()$ just above interface

$$\frac{\partial h}{\partial s} = \gamma h \quad \text{or} \quad \frac{\partial H}{\partial s} = \Gamma H \quad \text{at point II} \quad (10)$$

where γ may be a frequency-dependent constant. However, if the contact angle remains constant, we can show that (see Appendix C⁸)

$$\gamma = + \frac{1}{\sin \theta_c} \left\{ \left| \frac{z_{rr}}{(1+z_r^2)^{3/2}} \right|_{\text{II}} - \cos \theta_c \left| \frac{f_{rr}}{(1+f_r^2)^{3/2}} \right|_{\text{II}} \right\} \quad (10a)^9$$

$$\Gamma = \gamma a \quad (10b)$$

where

$$z_r = \frac{dz}{dr}, f_r = \frac{df}{dr}, z_{rr} = \frac{d^2z}{dr^2}, f_{rr} = \frac{d^2f}{dr^2}, \quad z \text{ on } W, f \text{ on } F$$

Method of Solution

We shall decompose ϕ into two parts, ϕ' and ϕ° . ϕ° is the velocity potential corresponding to a liquid contained by a rigid mean interface and the tank walls. Therefore, it satisfies the Laplace equation and the nonhomogeneous boundary condition on the contour, equation (8) for translation and equation (9) for pitching on F_c and W_c . It is noted that

$$\phi_{r^\circ} = \alpha_0 x \quad (11)$$

while ϕ_{r° can be constructed numerically.

ϕ' is the perturbed velocity potential due to sloshing which is governed by the Laplace equation, the zero normal velocity condition at the wall and the resultant interface condition. The first two are satisfied by expansion in normal modes and the last by equations (15) and (16). In this paper, the normal modes are determined by expansion into a set of auxiliary characteristic functions¹⁰ ψ_j , which is orthogonal on the curved interface, and satisfies the Laplace equation and the zero normal velocity wall condition¹¹. The interface condition governing normal modes is second order [see equation (7b)] subject to zero H at center and the contact point condition, equation (10a) at wall. The former is satisfied as ψ_j vanishes along tank center line. The interface equation is approximately satisfied by a modified Galerkin method (the strict Galerkin method is given in [19]), which is illustrated in Appendix B. In this method, the contact point condition was imposed on the whole series, not term by term. The numerical results shown later substantiate the method used.

The force and moment are obtained by integration of pressure, not only on the wall, but also on the interface since the direct surface tension force and moment on the tank is equivalent to those on the interface due to pressure, assuming the interface inertia is negligible, as well as the interface mass. To put results in the mechanical model form, the divergence theorem has been most useful (with some manipulations).

Analytical Results

Free Oscillations. For free oscillations, the natural mode ϕ_k is expanded into a truncated series of the auxiliary eigenfunctions, i.e.,

⁸ Appendix is not in order of mention.

⁹ An equivalent form was first derived in reference [15].

¹⁰ For direct application of the Winslow method [17], we impose the simpler normal derivation condition, $\partial\psi_j/\partial n_0 = \lambda\psi_j$, on F and used the well-known influence coefficient technique to determine the eigenvector ψ_j on the intersurface, the eigenvalue λ .

¹¹ Strictly speaking, the boundary conditions are imposed in Winslow method, while the solution not proven to satisfy these conditions is correct on physical grounds (see Appendix A).

$$\Phi_k = \frac{\phi_k}{\omega a^2} = \sum_{j=1}^{J_{mz}} c_{kj} \psi_{mj} \cos(m\theta), \quad \psi_j = \psi_{mj} \cos(m\theta) \quad (12a, b)$$

$$H_k = \frac{h_k}{a} = - \sum_{j=1}^{J_{mz}} c_{kj} \psi_{mj} \cos(m\theta) \quad (12c)$$

c_k is the k th eigenvector of the following matrix equation obtained by a modified Galerkin method (Appendix B) from integrating the nondimensional equation (7b) with $\theta_y = 0$ and weighting function $\psi_{mj} dS/\alpha_{mj}^2$

$$\left\{ \begin{aligned} & -\Gamma[\nu_{mj}] + [\gamma_{mj}] + m^2[\epsilon_{mj}] \\ & + \frac{\Delta\rho}{\rho} N_B[\beta_{mj}] - \Omega^2[\Delta_{mj}] \end{aligned} \right\} \{c_j\} = 0, \quad j = 1 \text{ to } J_{mz} \quad (13)$$

where

$$\beta_{mj} = \frac{\lambda_{mj}}{\alpha_{mj}^2} \int_F \frac{\psi_{mj} \psi_{mj} \epsilon_1}{\sqrt{1+F_R^2}} dS, \quad \epsilon_1 = \text{sgn} \left(\frac{\partial R}{\partial s} \right) \quad (13a)^{12}$$

$$\epsilon_{mj} = \frac{\lambda_{mj}}{\alpha_{mj}^2} \int_F \frac{1}{R^2} \psi_{mj} \psi_{mj} dS \quad (13b)$$

$$\nu_{mj} = \frac{2\pi\lambda_{mj}}{\alpha_{mj}^2} [R\psi_{mj}\psi_{mj}] \quad (13c)$$

$$\gamma_{mj} = \frac{\lambda_{mj}}{\alpha_{mj}^2} \left\{ \int_F [1+F_R^2]^{-1} \frac{d\psi_{mj}}{dR} \frac{d\psi_{mj}}{dR} dS - \int_F G\psi_{mj}\psi_{mj} dS \right\} \quad (13d)$$

$$\Delta_{mj} = \frac{1}{\alpha_{mj}^2} \int_F \psi_{mj} \psi_{mj} dS = \delta_{ij}, \quad (13e)$$

$$\alpha_{mj}^2 = \int_F \psi_{mj}^2 dS \quad (13f)$$

and $m = 1$ for lateral excitation of a rigid tank.

Forced Oscillations. Let

$$\phi' = -\omega a^2 \sum_{k=1}^{K_{mz}} d_k \Phi_k, \quad d_k = \frac{\bar{d}_k \Omega^2}{\Omega_k^2 - \Omega^2}, \quad \Phi_k = \sum_{j=1}^{J_{mz}} c_{kj} \psi_j \quad (14a, b, c)$$

in order to satisfy the interface condition that

$$\sum_{k=1}^{K_{mz}} d_k (\Omega_k^2 - \Omega^2) \Phi_k = -\Omega^2 \Phi^\circ + \epsilon_2 N_B \frac{\Delta\rho}{\rho} \frac{x}{a} \theta_y \equiv \Phi_N \Omega^2 \quad (15)$$

$\epsilon_2 = 0$ for translational oscillation, $\epsilon_2 = 1$ for pitching oscillation. We have by the Galerkin procedure [19, equation (115.5), p. 435]

$$\sum_{k=1}^{K_{mz}} \bar{d}_k \int_F \Phi_k (-H_l) dS = \int_F \Phi_N (-H_l) dS, \quad l = 1 \text{ to } K_{mz} \quad (16)$$

A weighting function of H_l was used taking advantage of the biorthogonal relation, equation (17). This assures the first K_{mz} components of the error as series in ϕ_k be zero. \bar{d}_k can be solved from equation (16) by matrix inversion. There is no need of storing information of ψ_j inside the fluid domain as only the force and moment are of interest. It is noted in the limit [8 and 9]

$$\int_F \Phi_k (-H_l) dS = \delta_{kl} \int_F \Phi_k (-H_k) dS \quad (17)^{13}$$

¹² The orthogonality property of ψ_j , thus, ψ_{mj} , can be easily proved [16] as in the high- G case $\cos^2(m\theta)$ in the integrand are uniformly omitted as they contribute the same factor, $1/2$.

¹³ This is referred to as biorthogonal relation.

then,

$$\bar{d}_i = \frac{\int_F \Phi_N(-H_i) dS}{\int_F \Phi_i(-H_i) dS} \quad (18)$$

which was utilized in proving that a unique spring-mass system exists for both pitching and translation

Force and Moment The force and moment exerted by a spring-mass system, Fig 2, without damping were written in the following form [20] and remain valid if contributions due to direct surface tension are included

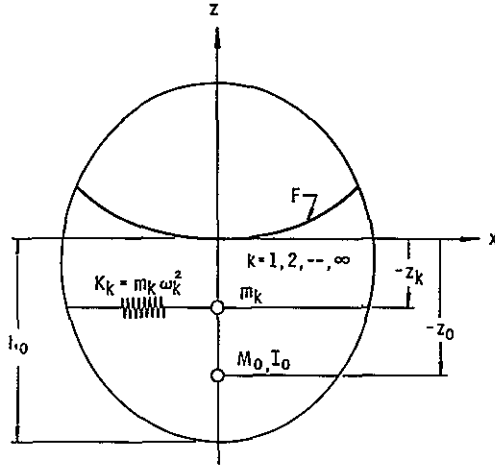


Fig 2 Equivalent mechanical model

$$F_H = F_z - M_F g \theta_y \quad (19)$$

$$F_{HT} = x_0 \omega^2 M_F \left\{ 1 + \sum_{k=1}^{\infty} \frac{m_k}{M_F} \frac{1}{\left(\frac{\omega_k^2}{\omega^2} - 1 \right)} \right\} \quad (20)$$

$$M_{vT} = x_0 \omega^2 M_F h_0 \left\{ \frac{z_{CG}}{h_0} + \sum_{k=1}^{\infty} \frac{m_k}{M_F} \left(\frac{z_k}{h_0} + \frac{g}{h_0 \omega^2} \right) \frac{1}{\left(\frac{\omega_k^2}{\omega^2} - 1 \right)} \right\} \quad (21)$$

$$F_{HP} = \theta_y \omega^2 M_F h_0 \left\{ \frac{z_{CG}}{h_0} + \sum_{k=1}^{\infty} \frac{m_k}{M_F} \left(\frac{z_k}{h_0} + \frac{g}{h_0 \omega^2} \right) \frac{1}{\left(\frac{\omega_k^2}{\omega^2} - 1 \right)} \right\} \quad (22)$$

$$M_{vP} = \theta_y \omega^2 M_F h_0^2 \left\{ \frac{I_F}{M_F h_0^2} + \sum_{k=1}^{\infty} \frac{m_k}{M_F} \left(\frac{z_k}{h_0} + \frac{g}{\omega^2 h_0} \right)^2 \frac{1}{\left(\frac{\omega_k^2}{\omega^2} - 1 \right)} \right\} + \theta_y g M_F h_0 \left(\frac{z_{CG}}{h_0} \right) \quad (23)$$

with rigid mass m_0 , its location z_0 , and moment of inertia I_0 given by

$$\frac{m_0}{M_F} = 1 - \sum_{k=1}^{\infty} \frac{m_k}{M_F} \quad (24)$$

$$\frac{z_0}{h_0} = \frac{1}{\frac{m_0}{M_F}} \left[\frac{z_{CG}}{h_0} - \sum_{k=1}^{\infty} \frac{z_k}{h_0} \frac{m_k}{M_F} \right] \quad (25)$$

$$\frac{I_0}{M_F h_0^2} = \frac{I_F}{M_F h_0^2} - \left(\frac{m_0}{M_F} \frac{z_0^2}{h_0^2} + \sum_{k=1}^{\infty} \frac{m_k}{M_F} \frac{z_k^2}{h_0^2} \right) \quad (26)$$

The force due to liquid pressure and direct surface tension is

$$F_z = \int_{W_F + F_e} p \frac{\partial x}{\partial n} dS \quad (27)^{14}$$

and the moment due to liquid pressure and direct surface tension is

$$M_v = \int_{W_F + F_e} p \left(z \frac{\partial x}{\partial n} - x \frac{\partial z}{\partial n} \right) dS \quad (28)$$

it can be shown that

$$\frac{m_k}{M_F} = \bar{d}_{kT} f_k' \frac{1}{V} \quad (29)$$

where

$$f_k' = \sum_{j=1}^{\infty} c_{kj} \int_F \lambda_j \psi_j \frac{x}{a} dS \quad (29a)$$

$$\bar{d}_{kT} = \frac{1}{\beta_k^2} \int_{W_F + F} \Phi_k \frac{\partial x}{\partial n} dS \cong \frac{1}{\beta_k^2} \sum_{j=1}^{J_{mx}} c_{kj} \lambda_j \int_F \frac{x}{a} \psi_j dS \quad (29b)^{15}$$

$$V = V_L / \rho a^2, \beta_k^2 = \int_F \Phi_k (-H_k) dA \quad (29c, d)$$

and

$$\frac{z_k}{a} = \frac{1}{\frac{m_k}{M_F}} \left(\frac{l_k'}{V} \right) \quad (30)$$

where

$$l_k' = \bar{d}_{kT} \sum_{j=1}^{\infty} c_{kj} \mu_j \quad (30a)$$

$$\mu_j = \int_{F+W} \psi_j \left(\frac{z}{a} \frac{\partial x}{\partial n} - \frac{x}{a} \frac{\partial z}{\partial n} \right) dS \quad (30b)$$

and that

$$I_F = M_F a^2 I_F^*, I_F^* \cong \frac{1}{V} \int_{W_F + F} \Phi_p \left(\frac{z}{a} \frac{\partial r}{\partial n} - \frac{x}{a} \frac{\partial z}{\partial n} \right) dS \quad (31a, b)$$

In deriving the mechanical model, ρ_u has been set to zero. A simple modification can be made for small ullage density by using

$$N_{Be} = \frac{\Delta p}{\rho} N_B$$

the effective Bond number instead of the Bond number based on the density of the liquid, provided that the dynamic pressure due

¹⁴ It is important to note that great simplification in results, when reduced to the mechanical model for both translation and pitching, is achieved by consideration of balance of forces acting on the thin interface as a free body. The force and moment on the tank due to direct surface tension must be equal to those acting on the interface by liquid pressure. These were not included in reference [15].

¹⁵ For finite J_{mx} , it was found that \bar{d}_{kT} , determined by matrix inversion of equation (16) without using biorthogonal relation, yields results in better agreement with Dodge's theory [12] than equation (29b) which is correct in the limit.

to ullage motion is negligible

It is noted that to prove the mechanical model for pitching motion frequent application of divergence theorem combined with differential properties of the auxiliary characteristic functions and the coordinate functions x and z were used. For brevity, the details are not included in this paper.

Numerical Examples The present computer program has been checked out for several examples in Appendix III. These include cylindrical tanks, spherical tanks, and spheroidal tanks. Good agreement of first spring-mass with known theory or experiments was obtained with as few as 11 or 12 interface net points and the first seven auxiliary characteristic functions. Finer nets may yield higher accuracies, especially for the higher modes.

Machine Time For a 12×18 mesh and a 12×12 mesh, the CDC-6600 central process time is a little over 2 min, while for the 23×34 mesh it is about 21 min. Most of the computing time was expended for the generation of influence coefficients, each of which is a Neumann problem. However, the influence coefficient method may be more convenient than the inversion of a large matrix, if not faster. No computer running time was reported in references [14 and 15], which finds natural modes by finite-differences and (partial) matrix inversion.

Conclusion

It seems that the present method yielded a practical way of computing the fundamental natural frequency, the first slosh mass, and its location. Higher masses and locations are usually not needed for design purposes and can be obtained by using finer meshes and longer machine time. A computer program utilizing triangular meshes and Winslow method [17] has been successfully employed and is expected to be completed in the near future for the titled problem. However, the present logical diagram may be limited to a convex axisymmetric tank for good accuracy.

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References

1. Abramson, H. N., ed., "The Dynamic Behavior of Liquids in Moving Containers," NASA Office of Scientific and Technical Information SP-106, Washington, 1966.
2. Abramson, H. N., "Dynamic Behavior of Liquids in Moving Containers," *Applied Mechanics Reviews*, Vol. 16, No. 7, July 1963, pp. 501-506.
3. Cooper, R. M., "Dynamics of Liquids in Moving Containers," *ARS Journal*, Vol. 30, Aug. 1960, p. 725.
4. Neu, J. T., and Good, R. J., "Equilibrium Behavior of Fluids in Containers at Zero-Gravity," *AIAA Journal*, Vol. 1, No. 4, Apr. 1963, pp. 814-819.
5. Satterlee, H. M., and Chun, J. H., "Meniscus Shape Under Reduced-Gravity Conditions," Symposium on Fluid Mechanics and Heat Transfer Under Low Gravitational Conditions, June 1965.
6. Hastings, L. J., and Rutherford, R., III, "Low Gravity Liquid-Vapor Interface Shapes in Axisymmetric Containers and a Computer Solution," NASA TM X-53790, Oct. 1968.
7. Petrish, D. A., and Otto, E. W., "Studies of the Liquid Vapor Interface Configuration in Weightlessness," ARS Space Power System Conference, Santa Monica, Calif., Paper No. 2514-62, 1962.
8. Satterlee, H. M., and Reynolds, W. C., "The Dynamics of the Free Liquid Surface in Cylindrical Containers Under Strong Capillary and Weak Gravity Conditions," TR LG-2, Department of Mechanical Engineering, Stanford University, May 1, 1964.
9. Yeh, G. C. K., "Free and Forced Oscillations of a Liquid in an Axisymmetric Tank at Low-Gravity Environments," *JOURNAL OF APPLIED MECHANICS*, Vol. 34, No. 1, TRANS ASME, Vol. 89, Series E, Mar. 1967, pp. 23-28.
10. Dodge, F. T., and Garza, L. R., "Experimental and Theoretical Studies of Liquid Sloshing at Simulated Low Gravities," Technical Report No. 2, Contract No. NAS8-20290, Control No. DCN 1-6-75-00010, SwRI Project 02-1846, Southwest Research Institute, Oct. 1966, see also *JOURNAL OF APPLIED MECHANICS*, Vol. 34, No. 3, TRANS ASME, Vol. 89, Series E, Sept. 1967, pp. 555-562.
11. Dodge, F. T., and Garza, L. R., "Simulated Low-Gravity Sloshing in a Cylindrical Tank Including Effects of Damping and Small Liquid Depth," *Proceedings, 1968 Heat Transfer and Fluid Mechanics Institute*, Stanford University Press, 1968, pp. 67-69.
12. Dodge, F. T., and Garza, L. R., "Simulated Low-Gravity Sloshing in Spherical Tanks and Cylindrical Tanks With Inverted Ellipsoidal Bottoms," Technical Report No. 6, Contract NAS8-20290, Control No. DCN 1-6-75-00010, SwRI Project No. 02-1846, Southwest Research Institute, Feb. 1968.
13. Dodge, F. T., and Garza, L. R., "Slosh Force, Natural Frequency, and Damping of Low-Gravity Sloshing in Oblated Ellipsoidal Tanks," Technical Report No. 7, Contract No. NAS8-20290, Control No. DCN 8-75-00043(1F), SwRI Project No. 02-1846, Southwest Research Institute, Feb. 1969.
14. Concus, P., Crane, G. E., and Satterlee, H. M., "Low Gravity Lateral Sloshing in Hemispherically Bottomed Cylindrical Tanks," *Proceedings, 1968 Heat Transfer and Fluid Mechanics Institute*, Stanford University Press, 1968, pp. 80-97.
15. Concus, P., Crane, G. E., and Satterlee, H. M., "Small Amplitude Lateral Sloshing in Spheroidal Containers Under Low Gravity Conditions," NASA CR-72500, LMSC-A944673, Lockheed Missiles and Space Co., Sunnyvale, Calif., February 4, 1969.
16. Chu, W. H., "Low Gravity Liquid Sloshing in an Arbitrary Axisymmetric Tank Performing Translational Oscillations," Technical Report No. 4, Contract No. NAS8-20290, Control No. DCN 1-6-75-00010, SwRI Project No. 02-1846, Southwest Research Institute, Mar. 1967.
17. Winslow, A. M., "Numerical Solutions of the Quasilinear Poisson Equations in a Nonuniform Triangular Mesh," *Journal of Computational Physics*, Vol. 1, No. 2, Nov. 1966, pp. 149-172.
18. Wang, C. T., *Applied Elasticity*, McGraw-Hill, New York, 1953.
19. Sokolnikoff, I. S., *Mathematical Theory of Elasticity*, 2nd ed. McGraw-Hill, New York, 1956.
20. Abramson, H. N., Chu, W. H., and Ransleben, G. E., Jr., "Representation of Fuel Sloshing in Cylindrical Tanks by an Equivalent Mechanical Model," *ARS Journal*, Dec. 1961, pp. 1697-1705.

APPENDIX A

Brief Description of a Computer Program

The following steps of a computer program are briefly described.

Construction of a Triangular Mesh The triangular mesh is generated as described in reference [17] except that a simple parallelogram is used as the logical diagram, Fig. 3. For a cylindrical tank at Bond number 100, the physical diagram is shown in Fig.

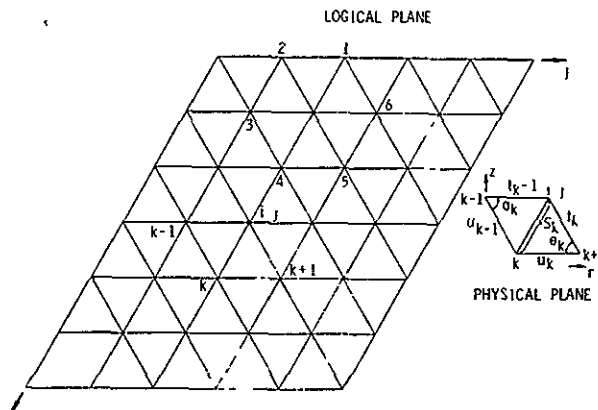


Fig. 3 Simple logical diagram for triangular mesh

4(a) For a spheroidal tank ($c = 0.5$) at Bond number 5, a triangular mesh is shown in Fig 4(b). The lengths of the edge of the parallelogram can be adjusted for each individual case to yield desired triangular meshes. A continuous wall needs to be broken into two parts for the logical diagram. This only affects the local distribution of the triangular mesh and has shown to yield good results for a spheroidal tank, as well as a cylindrical tank.

Construction of Auxiliary Characteristic Functions The characteristic functions ψ satisfy

$$\nabla^2 \psi = 0 \quad (32)$$

$$\frac{\partial \psi}{\partial n_0} = 0 \quad \text{on } W \quad (33)$$

$$\frac{\partial \psi}{\partial n_0} = \lambda \psi \quad \text{on } F \quad (34)$$

ψ can be solved numerically with the constructed triangular mesh by Winslow method [17]. Contact point is treated as one of the mesh points, as are the other boundary points¹⁶. Hence, $\partial \psi / \partial n$ may be discontinuous at the contact point. Zero contact

angle cannot be constructed graphically but results of decrease mesh size give closer and closer approximations to the interface and would probably lead to the correct limiting value.

For an interior joint, v_j , [$\psi = \psi_{1,j}$, $\psi_k = \psi_k(z, j)$, $r_k = r_k(z, j)$, $r = r_{1,j}$]

$$\sum_{k=1}^6 \omega_k(\psi_k - \psi) - \frac{m^2}{r_{1,j}} A_{1,j} \psi = 0 \quad (35)$$

where

$A_{1,j}$ = area of the v_j th dodecagon [17] inside the fluid domain
 $r_{1,j}$ = radius of the v_j th point

$$\omega_k = \frac{1}{2} (\lambda_k \bar{r}_k \cot \theta_k + \lambda_{k-1} \bar{r}_{k-1} \cot \sigma_k) \quad k = 1 \text{ to } 6 \quad (35a)^{17}$$

$$\bar{r}_k = \frac{1}{2} (r_{1,j} + r_k + r_{k+1}) \quad \lambda_k = 1 \quad (35b, c)$$

θ_k , σ_k , Fig 3, can be expressed in terms of l_k , u_{k+1} , l_{k-1} , u_{k-1} , and s_k .

For interface point,

$$\sum_{k=1}^6 \omega_k(\psi_k - \psi) - \frac{m^2}{r_{1,j}} A_{1,j} \psi + \left(\frac{\partial \psi}{\partial n} \right)_{1,j} \left[\frac{1}{2} s_3 + \frac{1}{2} s_6 \right] r_{1,j} = 0 \quad (36)$$

where

$$\lambda_6 = \lambda_1 = \lambda_2 = 0, \quad \lambda_3 = \lambda_4 = \lambda_5 = 1$$

To solve for the eigenfunctions on the interface, we use the influence coefficient method in which $(\partial \psi / \partial n)_{1,j} = 0$ except $(\partial \psi / \partial n)_{1,j} = 1$ for the j th column of the influence matrix. A standard eigenvalue problem involving only the interface points, excluding $\psi_{1,1}$ at $r = 0$, is needed to obtain the eigenvalues λ_j and eigenvectors ψ_j . Knowing the j th eigenvector on the interface, the corresponding value of ψ_j on the wall can be easily solved numerically again by the method of overrelaxation.

For v_j th point on the tank wall

$$\sum_{k=1}^6 \omega_k(\psi_k - \psi) - \frac{m^2}{r_{1,j}} A_{1,j} \psi = 0 \quad (37)$$

$\lambda_3 = \lambda_4 = \lambda_5 = 0$ and $\lambda_1 = \lambda_2 = \lambda_6 = 1$ on the bottom wall

$\lambda_4 = \lambda_5 = \lambda_6 = 0$ and $\lambda_1 = \lambda_2 = \lambda_3 = 1$ on the side wall

On center line, $r = 0$,

$$\psi = 0 \quad \text{for } m \geq 1$$

$$\frac{\partial \psi}{\partial r} = 0 \quad \text{for } m = 0 \quad (38)$$

$$\lambda_1 = \lambda_2 = \lambda_3 = 0 \text{ and } \lambda_4 = \lambda_5 = \lambda_6 = 1$$

At contact point $z = 1$, $j = j_{mz}$

$$\sum_{k=1}^6 \omega_k(\psi_k - \psi) - \frac{m^2}{r_{1,j}} A_{1,j} \psi + \left(\frac{\partial \psi}{\partial n} \right)_{1,j_{mz}} \left(\frac{1}{2} s_3 \right) r_{1,j} = 0 \quad (39)$$

$$\lambda_3 = 1, \lambda_1 = \lambda_2 = \lambda_4 = \lambda_5 = \lambda_6 = 0$$

Calculation of Interface Shape A program based on the theory of reference [6] was written to generate the interface shape with desired net points for a given empty fraction or central depth and a

¹⁶ This represents conservation of mass in the triangle containing contact point. Experience shows that the contact point condition should not be imposed on ψ .

¹⁷ Note $(\lambda_{j-1/2})$ reference [17] = λ_{j-1} , $(\lambda_{j+1/2})$ reference [17] = λ_j .

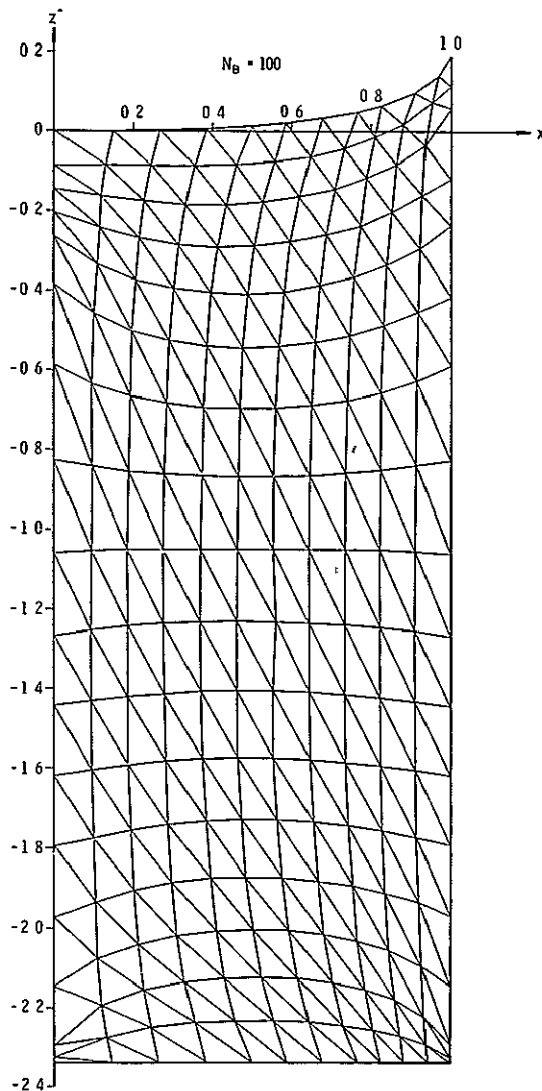


Fig 4(a) Physical diagram of triangular mesh—cylindrical tank

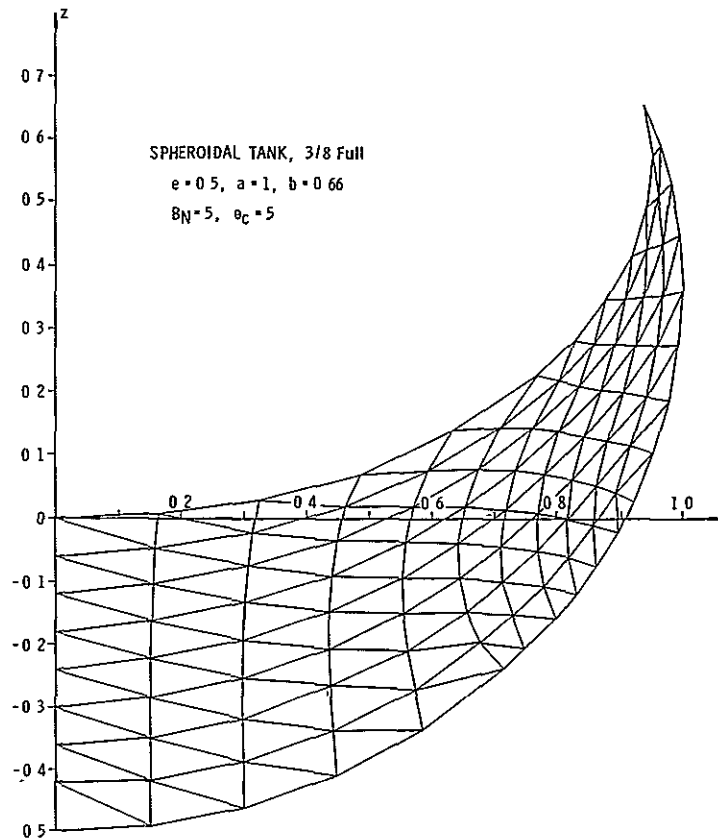


Fig 4(b) Triangular mesh for spheroidal tank at low gravity

given contact angle. This program also calculates contact point constant Γ but limits the contact angle to possibly 2 deg or greater.

Calculation of Natural Frequencies, Slosh Masses, and Their Location
The remaining steps are relatively routine and, therefore, will not be described. It is, however, remarked that trapezoidal rule and midpoint rules were employed conveniently in evaluating the integrals. For some quantities, quadratic fittings were made, such as dF/dR , before entering the quadrature formulas.

APPENDIX B

A Modified Galerkin Method

In the Galerkin method, it is generally assumed [19] that each of the coordinate functions of a complete set satisfies the same boundary conditions as the exact solution. This condition seems to be a sufficient condition rather than a necessary condition since the series expansion as a whole may satisfy the prescribed conditions, not necessarily term by term. Since the solution, if continuous, can be expanded into a convergent series containing the complete set, the failure of the error minimization process, if it occurs, must lie in the insufficient differentiability of the series. Therefore, the present method as applied to a second-order ordinary differential equation performs integration by part, then imposes the remaining boundary condition¹⁵ as illustrated by the following example.

Let y be governed by the simple equation

¹⁵ This may be a new technique to satisfy the boundary condition or conditions. However, a general proof and extension remain to be done.

$$\frac{d^2y}{dx^2} + \lambda^2 y = 0, \quad \lambda \neq \frac{2m-1}{2} \pi \quad (40)$$

subject to

$$y = 0 \quad \text{at } x = 0 \quad (41)$$

$$\frac{dy}{dx} = \Gamma_1 \quad \text{at } x = 1 \quad (42)$$

We shall select a complete set in $(0, 1)$ which satisfies $y = 0$ at $x = 0$ but $y \neq 0$ at $x = 1$. To be specific, we express

$$y = \sum_{m=1}^{\infty} C_m \sin(\lambda_m x) \quad (43)$$

where

$$\lambda_m = \frac{2m-1}{2} \pi$$

Then, the modified Galerkin procedure minimizes the error of the differential equation as follows

$$\begin{aligned} 0 &= \int_0^1 \sin(\lambda_n x) \left[\frac{d^2y}{dx^2} + \lambda^2 y \right] dx \\ &= \left[\sin(\lambda_n x) \frac{dy}{dx} \right]_0^1 - \int_0^1 \lambda_n \frac{dy}{dx} \cos(\lambda_n x) dx \\ &\quad + \int_0^1 \lambda^2 y \sin(\lambda_n x) dx = (-1)^{n-1} \Gamma_1 \\ &\quad - \int_0^1 \lambda_n \lambda_m \sum_{m=1}^{\infty} C_m \cos(\lambda_n x) \cos(\lambda_m x) dx \end{aligned}$$

APPENDIX III
NUMERICAL EXAMPLES

A. Flat Interface with High Bond Number in a Cylindrical Tank

$N_B = 1000$, $\frac{h_0}{a} = 2.34$, $a = 0.68$ with $\Gamma = 0$, a 12×18 mesh yielded $\frac{\omega_1^2 a}{g} = 1.85$ compared with 1.847 from exact theory (Ref. 1 of Appendix B).

$\frac{m_1}{m_F} = 0.193$ compared with 0.194 from high-G theory (Ref. 19 of Appendix B).

$z_1 = -0.729$ in. compared with -0.724 in. from high-G theory (Ref. 19 of Appendix B).

B. Flat Interface with Low Bond Number in a Cylindrical Tank $N_B = 10$, $\frac{h_0}{a} = 2.34$, $a = 0.68$ with $\Gamma = 0$, 12×18 mesh yielded $\frac{\omega_1^2 a}{g} = 2.15$ compared with 2.46 from exact theory. A finer mesh is required for better agreement.C. Curved Interface with Low Bond Number $N_B = 100$, $\frac{h_0}{a} = 2.34$, $a = 0.68$, $\theta_c = 2^\circ$ yielded $\frac{\omega_1^2 a}{g} = 1.860$, $\frac{m_1}{\rho a^3} = 0.442$, $z_1 \cong -0.724$ in. compared with theoretical value of $\frac{\omega_1^2 a}{g} = 1.777$, $\frac{m_1}{\rho a^3} = 0.438$, $z_1 \cong -0.73$ in. (Ref. 12 of Appendix B). It is noted that experimental value of $\frac{\omega_1^2 a}{g}$ for $\theta_c = 0^\circ$ is around 1.78 to 1.80. Finer net may lead to better approximation.D. Spherical Tank with Flat Interface $N_B = 1000$, $\frac{h_0}{a} = 1$, $a = 1$ with $\Gamma = 0$, 11×11 mesh yielded $\frac{\omega_1^2 a}{g} = 1.54$ compared with high-G sloshing value of $\frac{\omega_1^2 a}{g} = 1.54$.

- E. Spherical Tank with "Folding" Interface $N_B = 10$, $\frac{h_0}{a} \approx 1.14^\dagger$, $a = 1$, $\theta_c = 4.96^\circ$ ($\Gamma = -65.788$), 12×12 mesh yield $\frac{\omega_1^2 a}{g} = 1.469$, $\frac{m_1}{\rho a^3} = 0.604$, $z_1 = -0.742$ in. compared with Lockheed result (Ref. 16 of Appendix B) of $\frac{\omega_1^2 a}{g} = 1.507$ (for $\theta_c = 5^\circ$). The first slosh mass is much larger than that of Lockheed, probably due to the inclusion of force directly attributed to surface tension in the present theory which yielded confirmed results for cylindrical tanks (Case 3). The value of z_1 would be the location of the center of the sphere if the contribution due to direct surface tension [i. e., the integral over F in Eq. (30b)] is neglected.
- F. Spheroidal tank with Folding Interface Ellipticity $e = 0.5$, major axis $a = 1$, minor axis $b = 0.866$, $N_B = 5$, $\beta = 0.6263$ (3/8 full tank), $\frac{h_0}{a} \approx 0.4999$, $\theta_c = 5.06^\circ$, ($\Gamma = -38.430$, $V_L = 1.35$). ‡ 12×9 mesh (non-uniform on free surface) yielded $\frac{\omega_1^2 a}{g} = 1.114^{\dagger\dagger}$, $\frac{m_1}{\rho V_L} = 0.582$, $z_1 = -0.5841$ compared with Lockheed result of $\frac{\omega_1^2 a}{g} = 0.966$. It is noted that the value of z_1 puts the first mass under the tank bottom which should be examined by future experiments when available.

† Calculated for 3/4-full tank.

‡ For $\theta_c = 5.06^\circ$ the Γ used in ASME paper APM-EEE was in error and the correct value is given here as are the results.

†† With 23×17 mesh, $\frac{\omega_1^2 a}{g} = 1.079$, $\frac{m_1}{\rho V_L} = 0.5736$, $z_1 = -0.4644$; and

$\frac{\omega_2^2 a}{g} = 15.03$ while Lockheed's result for $\frac{\omega_2^2 a}{g}$ is 13.26. It is uncertain which results are more accurate. The value of Γ has not been given by Lockheed to facilitate the explanation of the differences.

APPENDIX IV.

LISTING AND INPUT SAMPLE OF TRIPOT

```

PROGRAM TRIPOT(INPUT,OUTPUT,TAPE7,TAPE6)
C   NORMAL COORDINATES
C   02-1846-02 TRIANGULAR MESH GENERATOR AND SOLUTION OF
C   NORMAL MODES
COMMON C(40,24,8),PHI(40,24),WB,EPS,ITA,N,M,W0,RHO,R(6),YT(7)
DIMENSION X(40,24),Y(40,24),ITITLE(12)
DIMENSION ELAMDA(6),SS(6),T(6),CT(6),CS(6),W(6),XT(7),A(6)
DIMENSION CV(24),F(24,24),EVAL(24,24),EVEC(24,24),EVI(1,1)
DIMENSION EIGVAL(24),NVAL(24),PHIS(24,24),PHIW(64,24)
DIMENSION DFDR(24),DRS(24),DFRR(24),DS(24),DRW(64),DZW(64)
DIMENSION ALPJS(24),ENU(24,24),GJ(24),TKJ(24),RMX(24,24)
DIMENSION AMU(24),TALSK(24),COMS(24),RW(64),ZW(64),EKS(24),TK(24)
DIMENSION ELK(24),EMK(24),ZK(24),DELIJ(24,24),IROW(25),ICOL(25)
DIMENSION SIGK(24)
DIMENSION VNB(24),VNF(24),VNW(40),XK(24)
EQUIVALENCE (DFDR(1),C(1)),(DRS(1),C(25)),(DFRR(1),C(49)),
1 (DS(1),C(73)),(DRW(1),C(97)),(DZW(1),C(161)),(ALPJS(1),C(225)),
2 (ENU(1),C(249)),(GJ(1),C(825)),(TKJ(1),C(849)),(RMX(1),C(873)),
3 (AMU(1),C(1449)),(TALSK(1),C(1473)),(COMS(1),C(1497)),
4 (RW(1),C(1521)),(ZW(1),C(1585)),(EKS(1),C(1649)),(TK(1),C(1673)),
5 (ELK(1),C(1697)),(DELIJ(1),C(1721)),(IROW(1),C(2297)),
6 (ICOL(1),C(2322)),(SIGK(1),C(2347)),(EMK(1),C(2371)),
7 (ZK(1),C(2395)),(XK(1),C(2419))
KUN=6
LUN=7
NDIM=24
RHO=0.05
BETAC=0.5
W0=.01
PI=3.14159267
READ 94,ITITLE
94 FORMAT (12A6)
READ 95,M,N,ITA,IRS,WA,WB,EPS,RLGTH,BOND
95 FORMAT (4I5,5F10.0)
PRINT 96,ITITLE,M,N,ITA,WA,WB,EPS,IRS
96 FORMAT (26H1TRIANGULAR MESH GENERATOR/,1X,12A6/,
117H LOGICAL MESH IS ,I5,3H X ,I5/,32H MAXIMUM NUMBER OF ITERATIONS
2 IS,I5/,40H RELAXATION FOR LINEAR APPROXIMATION IS ,F7.3/,
353H INITIAL RELAXATION FACTOR FOR NONLINEAR SOLUTION IS ,F7.3/,
428H EPSILON FOR CONVERGENCE IS ,F10.6/,19H RESTART IS NUMBER ,I5/)
NM1=N-1
MM1=M-1
NM2=N-2
EM=1.0
MM=1
NPM1=N+M-1
IF (MM .EQ. 0) NM2=NM1
DO 97 J=2,NM1
DO 97 I=2,MM1
C(I,J,1)=0.5
C(I,J,2)=0.5
C(I,J,3)=0.5
97 CONTINUE
IF (IRS .EQ. 0) GO TO 3003
GO TO (3001,1061,3006,3008,3012,377,389),IRS
3003 CONTINUE
READ 200,(X(1,J),Y(1,J),J=1,N)
READ 200,(X(I,N),Y(I,N),I=2,MM1)
READ 200,(X(M,J),Y(M,J),J=1,N)
READ 200,(X(I,1),Y(I,1),I=2,MM1)
200 FORMAT (8F10.0)
DO 93 J=1,N
X(1,J)=X(1,J)/RLGTH

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      Y(1,J)=Y(1,J)/RLGTH
      X(M,J)=X(M,J)/RLGTH
93  Y(M,J)=Y(M,J)/RLGTH
      DO 92 I=2,MM1
      X(I,N)=X(I,N)/RLGTH
      Y(I,N)=Y(I,N)/RLGTH
      X(I,1)=X(I,1)/RLGTH
92  Y(I,1)=Y(I,1)/RLGTH
C   INTERPOLATE FROM BOUNDARIES TO GET FIRST GUESS
      DO 100 J=2,NM1
      DO 100 I=2,MM1
      X(I,J)=X(I,J-1)+X(I-1,J)-X(I-1,J-1)
      Y(I,J)=Y(I,J-1)+Y(I-1,J)-Y(I-1,J-1)
100  CONTINUE
      GO TO 3002
C   RESTART 1
3001 READ (LUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
      WRITE(KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
3002 CONTINUE
      IDIR=-1
      DO 103MOST=1,ITA
      SX=0.0
      KIT=MOST
      SY=0.0
      SRX=0.0
      SRY=0.0
      IDIR=-IDIR
      DO 102 K=2,MM1
      DO 102 L=2,NM1
      IF(IDIR) 1001,1002,1002
1001 I=MM1-K+2
      J=NM1-L+2
      GO TO 1003
1002 I=K
      J=L
1003 CONTINUE
      XTEMP=(X(I-1,J-1)+X(I+1,J+1)+X(I,J-1)+X(I,J+1)+X(I-1,J)+X(I+1,J)
1 )/6.0
      YTEMP=(Y(I-1,J-1)+Y(I+1,J+1)+Y(I,J-1)+Y(I,J+1)+Y(I-1,J)+Y(I+1,J)
1 )/6.0
      RX=(X(I,J)-XTEMP)*WA
      RY=(Y(I,J)-YTEMP)*WA
      X(I,J)=X(I,J)-RX
      Y(I,J)=Y(I,J)-RY
      SX=SX+X(I,J)**2
      SY=SY+Y(I,J)**2
      SRX=SRX+RX**2
      SRY=SRY+RY**2
102  CONTINUE
      IF (MOST-20*(MOST/20)) 1022,1021,1022
1021 WRITE (KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
      REWIND KUN
      IRS=1
      IF(MOST .EQ. 20) PRINT 410,IRS
410  FORMAT(21H0YOU MAY USE RESTART ,I4)
1022 CONTINUE
      SRX=SQRT(SRX/SX)
      SRY=SQRT(SRY/SY)
      IF (SRX .LE. EPS .AND. SRY .LE. EPS) 105,103
103  CONTINUE
      PRINT 201,SRX,SRY
201  FORMAT(51H1PROCESS DID NOT CONVERGE FOR INITIAL APPROXIMATION/,
110H EPS-X IS ,E15.7,12H EPS-Y IS ,E15.7//,41H X AND Y VALUES ARE
2 PRINTED BELOW BY ROWS/)
      DO 104 I=1,M
104  PRINT 202,(X(I,J),Y(I,J),J=1,N)

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202 FORMAT (8E15.7)
WRITE (KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
REWIND KUN
IRS=1
PRINT 410,IRS
STOP
105 PRINT 203,KIT
203 FORMAT (58H1PROCESS CONVERGED FOR INITIAL APPROXIMATION ON ITERATI
10N ,15/,33H X AND Y ARE PRINTED BELOW BY ROW/)
DO 106 I=1,M
PRINT 202,(X(I,J),Y(I,J),J=1,N)
106 CONTINUE
WRITE (KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
REWIND KUN
IRS=2
PRINT 410,IRS
GO TO 1062
C
RESTART 2
1061 READ (LUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
WRITE (KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
1062 CONTINUE
WX=WB
WY=WB
IDIR=-1
DO 135 MOST=1,ITA
IDIR=-IDIR
KIT=MOST
SRX=0.0
SRY=0.0
SX=0.0
SY=0.0
DO 110 I=2,MM1
DO 110 J=2,NM1
DXDX=((X(I-1,J-1)+2.0*X(I-1,J)+X(I,J+1))-(X(I,J-1)+2.0*X(I+1,J)
1 +X(I+1,J+1)))/6.0
DYDX=((Y(I-1,J-1)+2.0*Y(I-1,J)+Y(I,J+1))-(Y(I,J-1)+2.0*Y(I+1,J)
1 +Y(I+1,J+1)))/6.0
GAMMA=DXDX**2+DYDX**2
DXDY=((X(I-1,J)+2.0*X(I,J+1)+X(I+1,J+1))-(X(I-1,J-1)+2.0*X(I,J-1)
1 +X(I+1,J)))/6.0
DYDY=((Y(I-1,J)+2.0*Y(I,J+1)+Y(I+1,J+1))-(Y(I-1,J-1)+2.0*Y(I,J-1)
1 +Y(I+1,J)))/6.0
ALPHA=DXDY**2+DYDY**2
BETA=DXDY*DXDX+DYDX*DYDY
CP1=ALPHA-BETA
CP2=BETA
CP3=GAMMA-BETA
CP1=BETAC*CP1+(1.0-BETAC)*C(I,J,1)
CP2=BETAC*CP2+(1.0-BETAC)*C(I,J,2)
CP3=BETAC*CP3+(1.0-BETAC)*C(I,J,3)
C(I,J,1)=CP1
C(I,J,2)=CP2
C(I,J,3)=CP3
110 CONTINUE
DO 120 K=2,MM1
DO 120 L=2,NM1
IF (IDIR) 1011,1012,1012
1011 I=MM1-K+2
J=NM1-L+2
GO TO 1013
1012 I=K
J=L
1013 CONTINUE

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C1=C(I,J,1)
C2=C(I,J,2)
C3=C(I,J,3)
C4=C1
C5=C2
C6=C3
SUMC=C1+C2+C3+C4+C5+C6
IF (ABS(SUMC) - 1.0E-10) 120,120,111
111 XTEMP=(C1*X(I-1,J)+C2*X(I-1,J-1)+C3*X(I,J-1)+C4*X(I+1,J)
1 +C5*X(I+1,J+1)+C6*X(I,J+1))/SUMC
YTEMP=(C1*Y(I-1,J)+C2*Y(I-1,J-1)+C3*Y(I,J-1)+C4*Y(I+1,J)
1 +C5*Y(I+1,J+1)+C6*Y(I,J+1))/SUMC
RX=(X(I,J)-XTEMP)*WX
RY=(Y(I,J)-YTEMP)*WY
X(I,J)=X(I,J)-RX
Y(I,J)=Y(I,J)-RY
SRX=SRX+RX**2
SRY=SRY+RY**2
SX=SX+X(I,J)**2
SY=SY+Y(I,J)**2
120 CONTINUE
IF (MOST-20*(MOST/20)) 3005,3004,3005
3004 WRITE (KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
REWIND KUN
IRS=2
IF(MOST .EQ. 20) PRINT 410,IRS
3005 CONTINUE
IF (KIT-1) 126,126,127
126 SPRX=SRX
SPRY=SRY
127 EPSCX=SQRT(SRX/SX)
EPSCY=SQRT(SRY/SY)
IF (EPSCX .LE. EPS .AND. EPSCY .LE. EPS) GO TO 140
ETAX=SQRT(SRX/SPRX)
ETAY=SQRT(SRY/SPRY)
ELX=(WX+ETAX-1.0)/(WX*SQRT(ETAX))
ELY=(WY+ETAY-1.0)/(WY*SQRT(ETAY))
IF (ABS(ELX)-1.0) 129,129,128
128 WAX=WX
GO TO 130
129 WAX=2.0/(1.0+SQRT(1.0-ELX**2))-W0
130 IF (ABS(ELY)-1.0) 132,132,131
131 WAY=WY
GO TO 133
132 WAY=2.0/(1.0+SQRT(1.0-ELY**2))-W0
133 WX=RHO*WAX+(1.0-RHO)*WX
WY=RHO*WAY+(1.0-RHO)*WY
SPRX=SRX
SPRY=SPRY
135 CONTINUE
PRINT 204,KIT,EPSCX,EPSCY
204 FORMAT (43H1NONLINEAR SOLUTION DID NOT CONVERGE AFTER ,I5,11H ITER
1ATIONS/,10H EPS-X IS ,E15.7,12H EPS-Y IS ,E15.7/,34H X AND Y ARE
2 PRINTED BELOW BY ROWS/)
DO 136 I=1,M
PRINT 202,(X(I,J),Y(I,J),J=1,N)
136 CONTINUE
WRITE (KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
IRS=2
PRINT 410,IRS
STOP
140 PRINT 205,KIT
205 FORMAT (42H1NONLINEAR SOLUTION CONVERGED ON ITERATION,I5/,
134H X AND Y ARE PRINTED BELOW BY ROWS/)

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      DO 141 I=1,M
      PRINT 202,(X(I,J),Y(I,J),J=1,N)
141  CONTINUE
      WRITE (KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
      1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
      IRS=3
      PRINT 410,IRS
      GO TO 3007
C     RESTART 3
3006 READ (LUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
      1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
      WRITE (KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
      1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
3007 CONTINUE
      DO 301 J=1,N
      DO 301 I=1,M
      DO 300 K=1,6
      C(I,J,K)=1.0
300  CONTINUE
      C(I,J,7)=0.0
301  CONTINUE
      DO 302 J=1,N
      C(1,J,1)=0.0
      C(1,J,2)=0.0
      C(1,J,6)=0.0
      C(M,J,3)=0.0
      C(M,J,4)=0.0
302  C(M,J,5)=0.0
      DO 303 I=1,M
      C(I,1,1)=0.0
      C(I,1,2)=0.0
      C(I,1,3)=0.0
      C(I,N,4)=0.0
      C(I,N,5)=0.0
303  C(I,N,6)=0.0
      DO 350 I=1,M
      DO 350 J=1,N
      DO 305 K=1,6
      A(K)=0.0
      R(K)=0.0
      SS(K)=1.0
      T(K)=1.0
      CT(K)=0.0
      CS(K)=0.0
      ELAMDA(K)=C(I,J,K)
      XT(K)=0.0
305  YT(K)=0.0
      XTEMP=X(I,J)
      YTEMP=Y(I,J)
      IF (I-1) 307,307,306
306  XI(1)=X(I-1,J)
      YI(1)=Y(I-1,J)
307  IF (I-1) 308,309,309
308  XI(4)=X(I+1,J)
      YI(4)=Y(I+1,J)
309  IF (J-1) 311,311,310
310  XT(3)=X(I,J-1)
      YT(3)=Y(I,J-1)
311  IF (J-N) 312,313,313
312  XT(6)=X(I,J+1)
      YT(6)=Y(I,J+1)
313  IF (I-1) 316,316,314
314  IF (J-1) 316,316,315
315  XT(2)=X(I-1,J-1)
      YT(2)=Y(I-1,J-1)
316  IF (I-M) 317,319,319

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317 IF (J-N) 318,319,319
318 XT(5)=X(I+1,J+1)
    YT(5)=Y(I+1,J+1)
319 CONTINUE
    XT(7)=XT(1)
    YT(7)=YT(1)
    DO 323 K=1,6
320 SS(K)=SQRT((XT(K)-XTEMP)**2+(YT(K)-YTEMP)**2)
    IF (SS(K) .EQ. 0) SS(K)=1.0
    IF (K-6) 321,322,322
321 T(K)=SQRT((XT(K+1)-XT(K))**2+(YT(K+1)-YT(K))**2)
    IF (T(K) .EQ. 0) T(K)=1.0
    R(K)=(XTEMP+XT(K)+XT(K+1))/3.0
    GO TO 323
322 T(K)=SQRT((XT(6)-XT(1))**2+(YT(6)-YT(1))**2)
    IF (T(K) .EQ. 0) T(K)=1.0
    R(K)=(XTEMP+XT(6)+XT(1))/3.0
323 CONTINUE
    DO 340 K=1,6
    IF (K-1) 324,324,326
324 IF (ELAMDA(6)) 325,331,325
325 COSS=(T(6)**2+SS(6)**2-SS(1)**2)/(2.0*T(6)*SS(6))
    GO TO 328
326 IF (ELAMDA(K-1)) 327,331,327
327 COSS=(T(K-1)**2+SS(K-1)**2-SS(K)**2)/(2.0*T(K-1)*SS(K-1))
328 CS(K)=1.0-COSS**2
    IF (CS(K)) 329,329,330
329 CS(K)=0.0001
330 CS(K)=COSS/SQRT(CS(K))
331 IF (ELAMDA(K)) 332,340,332
332 XA=0.5*(XT(K)+XTEMP)
    XB=(XT(K)+XT(K+1)+XTEMP)/3.0
    XC=0.5*(XT(K+1)+XTEMP)
    YA=0.5*(YT(K)+YTEMP)
    YB=(YT(K)+YT(K+1)+YTEMP)/3.0
    YC=0.5*(YT(K+1)+YTEMP)
    AB=SQRT((XA-XB)**2+(YA-YB)**2)
    BC=SQRT((YB-YC)**2+(XB-XC)**2)
    AD=SQRT((XA-XTEMP)**2+(YA-YTEMP)**2)
    BD=SQRT((XB-XTEMP)**2+(YB-YTEMP)**2)
    CD=SQRT((XC-XTEMP)**2+(YC-YTEMP)**2)
    ARA=0.5*(BC+CD+BD)
    ARA=SQRT(ARA*(ARA-BC)*(ARA-CD)*(ARA-BD))
    ARB=0.5*(AB+BD+AD)
    ARB=SQRT(ARB*(ARB-AB)*(ARB-BD)*(ARB-AD))
    A(K)=ARA+ARB
    IF (K-6) 334,333,333
333 COST=(T(6)**2+SS(1)**2-SS(6)**2)/(2.0*T(6)*SS(1))
    GO TO 335
334 COST=(T(K)**2+SS(K+1)**2-SS(K)**2)/(2.0*T(K)*SS(K+1))
335 CT(K)=1.0-COST**2
    IF (CT(K)) 336,336,337
336 CT(K)=0.0001
337 CT(K)= COST/SQRT(CT(K))
340 CONTINUE
    K=1
    IF (I-1) 341,341,342
341 IF ( J .EQ. 1) SS(3)=0.0
    IF ( J .EQ. N) SS(6)=0.0
    CV(J)=(.5*(SS(3)+SS(6)))*XTEMP
342 DO 343 K=2,6
    W(K)=ELAMDA(K)*R(K)*CT(K)+ELAMDA(K-1)*R(K-1)*CS(K)
343 W(K)=0.5*W(K)
    W(1)=ELAMDA(1)*R(1)*CT(1)+ELAMDA(6)*R(6)*CS(1)
    W(1)=0.5*W(1)
    IF (XTEMP) 345,344,345

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344 EMR=0.0
    GO TO 346
345 EMR=EM*EM/XTEMP
346 SUMC=0.0
    DO 347 K=1,6
      C(I,J,K)=W(K)
347 SUMC=SUMC+W(K)+A(K)*EMR
    C(I,J,8)=SUMC
    IF (J-1) 348,348,350
348 IF (MM) 349,350,349
349 C(I,J,1)=0.0
    C(I,J,2)=0.0
    C(I,J,3)=0.0
    C(I,J,4)=0.0
    C(I,J,5)=0.0
    C(I,J,6)=0.0
    C(I,J,8)=1.0
350 CONTINUE
C   SET COEFFICIENTS FOR CONTACT POINT
    C(1,N,1)=0.0
    C(1,N,2)=0.0
    C(1,N,5)=0.0
    C(1,N,6)=0.0
    READ 200,GFLG,GAMMA,THETAC
    THETAC=.0174532925*THETAC
    R(1)=X(1,N-2)
    R(2)=X(1,N-1)
    R(3)=X(1,N)
    YT(1)=Y(1,N-2)
    YT(2)=Y(1,N-1)
    YT(3)=Y(1,N)
    CALL SFIT(FR,FRR,XTEMP,YTEMP)
C   IF GFLG IS 0 COMPUTE GAMMA
    IF (GFLG) 3508,3503,3508
3503 R(3)=X(1,N)
    R(2)=X(2,N)
    R(1)=X(3,N)
    YT(3)=Y(1,N)
    YT(2)=Y(2,N)
    YT(1)=Y(3,N)
    CALL SFIT(ZR,ZRR,XTEMP,YTEMP)
    ZTEMP=COS(THETAC)
    IF (ABS(ZTEMP).GE.1.0) GO TO 3506
    GAMMA=(ABS(ZRR)-ABS(FRR)*ZTEMP)/SIN(THETAC)
    GO TO 3508
3506 GAMMA=0.0
3508 CONTINUE
3509 CONTINUE
    PRINT 4000,GAMMA
4000 FORMAT(7H0GAMMA ,E15.8)
    WRITE (KUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
    II=1
    IRS=4
    PRINT 410,IRS
    NM1=N
    GO TO 3011
C   RESTART 4
3008 II=1
    NM1=N
    READ (LUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
    1,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
    WRITE(KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
    1,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
    READ (LUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
    WRITE(KUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
3009 READ (LUN) (F(J,II),J=1,NM1)

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      IF (EOF,LUN) 3011,3010
3010 II=II+1
      WRITE(KUN) (F(J,II),J=1,NM1)
      GO TO 3009
3011 CONTINUE
      DO 355 J=1,N
      DO 355 I=1,M
355 PHI(I,J)=0.0
      IF (MM .NE. 0) CV(1)=0.0
      DO 360 ICOUNT=II,NM1
      C(1,ICOUNT,7)=CV(ICOUNT)
      CALL SOREL(1)
      DO 356 J=1,NM1
      F(J,ICOUNT)=PHI(1,J)
356 CONTINUE
      WRITE (KUN) (F(J,ICOUNT),J=1,NM1)
      C(1,ICOUNT,7)=0.0
360 CONTINUE
      IRS=5
      NM1=N
      PRINT 410,IRS
      GO TO 3013
C   RESTART 5
3012 READ (LUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
      1,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
      NM1=N
      WRITE(KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
      1,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
      READ (LUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
      WRITE(KUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
      DO 3014 I=1,NM1
3014 READ (LUN) (F(J,I),J=1,NM1)
      WRITE(KUN) (F(J,I),J=1,NM1)
3013 CONTINUE
      DO 361 I=1,NM1
      DO 361 J=1,NM1
      EVAL(I,J)=F(I,J)
361 CONTINUE
      CALL EIGEN(NM1,EVAL,EVI,EVEC,50,3,0,1,NDIM)
      DO 372 I=1,NM1
      NVAL(I)=I
      IF (EVAL(I,I)) 370,370,371
370 EIGVAL(I)=10.0E+60
      GO TO 372
371 EIGVAL(I)=1.0/EVAL(I,I)
372 CONTINUE
      DO 375 I=1,NM1
      K=I
      DO 374 J=K,NM1
      IF (EIGVAL(I)-EIGVAL(J)) 374,374,373
373 XTEMP=EIGVAL(J)
      II=NVAL(J)
      EIGVAL(J)=EIGVAL(I)
      NVAL(J)=NVAL(I)
      EIGVAL(I)=XTEMP
      NVAL(I)=II
374 CONTINUE
375 CONTINUE
      PRINT 400,ITITLE
400 FORMAT (1H1,12A6/)
      PRINT 401
401 FORMAT (77H EIGENVALUES AND EIGENVECTORS ASSOCIATED WITH INFLUENCE
      1 COEFFICIENT SOLUTIONS)
      NM2=N
      DO 376 I=1,NM2
      PRINT 402,I,EIGVAL(I)

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402 FORMAT (6HMODE ,I3,14H  EIGENVALUE ,E15.8)
      K=NVAL(I)
      PRINT 202,(EVEC(J,K),J=1,NM1)
376 CONTINUE
      WRITE (KUN) (EIGVAL(I),I=1,NM1),(NVAL(I),I=1,NM1),
1 ((EVEC(I,J),I=1,NM1),J=1,NM1),(NVAL(J),J=1,NM1)
      II=1
      IRS=6
      PRINT 410,IRS
      NM1=N
      NM2=N-1
      GO TO 382
C   RESTART 6
377 READ (LUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
      NM1=N
      NM2=N-1
      WRITE(KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
      READ (LUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
      WRITE(KUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
      DO 378 I=1,NM1
      READ (LUN) (F(J,I),J=1,NM1)
378 WRITE(KUN) (F(J,I),J=1,NM1)
      READ (LUN) (EIGVAL(I),I=1,NM1),(NVAL(I),I=1,NM1),
1 ((EVEC(I,J),I=1,NM1),J=1,NM1),(NVAL(J),J=1,NM1)
      WRITE (KUN) (EIGVAL(I),I=1,NM1),(NVAL(I),I=1,NM1),
1 ((EVEC(I,J),I=1,NM1),J=1,NM1),(NVAL(J),J=1,NM1)
379 READ (LUN) (PHIS(I,II),I=1,N),(PHIW(I,II),I=1,NPMM1)
      IF (EOF,LUN) 382,381
381 WRITE(KUN) (PHIS(I,II),I=1,N),(PHIW(I,II),I=1,NPMM1)
      II=II+1
      GO TO 379
382 CONTINUE
      DO 383 J=1,N
      DO 383 I=2,M
383 PHI(I,J)=0.0
      DO 388 ICOUNT=II,NM2
      L=NVAL(ICOUNT)
      DO 384 I=1,NM1
384 PHI(1,I)=EVEC(I,L)
      CALL SOREL(2)
      DO 385 J=1,N
385 PHIS(J,ICOUNT)=PHI(1,J)
      DO 386 J=1,N
386 PHIW(J,ICOUNT)=PHI(M,J)
      DO 387 K=1,NM1
      KPN=K+N
      L=M-K
387 PHIW(KPN,ICOUNT)=PHI(L,N)
      WRITE (KUN) (PHIS(I,ICOUNT),I=1,N),(PHIW(I,ICOUNT),I=1,NPMM1)
388 CONTINUE
      NM1=N
      NM2=N-1
      GO TO 391
C   RESTART 7
389 READ (LUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
      NM1=N
      NM2=N-1
      WRITE(KUN) ((X(I,J),Y(I,J),I=1,M),J=1,N)
1 ,(((C(I,J,K),I=1,M),J=1,N),K=1,3)
      READ (LUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
      WRITE(KUN) GAMMA,(((C(I,J,K),I=1,M),J=1,N),K=1,8),(CV(J),J=1,N)
      DO 390 I=1,NM1
      READ (LUN) (F(J,I),J=1,NM1)

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390 WRITE(KUN) (F(J,I),J=1,NM1)
   READ (LUN) (EIGVAL(I),I=1,NM1),(NVAL(I),I=1,NM1),
   1((EVEC(I,J),I=1,NM1),J=1,NM1),(NVAL(J),J=1,NM1)
   WRITE (KUN) (EIGVAL(I),I=1,NM1),(NVAL(I),I=1,NM1),
   1((EVEC(I,J),I=1,NM1),J=1,NM1),(NVAL(J),J=1,NM1)
   DO 392 J=1,NM2
   READ (LUN) (PHIS(I,J),I=1,N),(PHIW(I,J),I=1,NPMM1)
392 WRITE(KUN) (PHIS(I,J),I=1,N),(PHIW(I,J),I=1,NPMM1)
391 CONTINUE
   IRS=7
   PRINT 410,IRS
   NM1=N-1
   NM2=N-2
C   READ OTHER INPUTS
   READ 4002,ZCG,VOL,DEN,RHOU
4002 FORMAT (5E15.8)
   PRINT 400,ITITLE
   PRINT 4003,BOND,ZCG,VOL,DEN,RHOU
4003 FORMAT(13H0BOND NUMBER ,E15.8, 7H   ZCG ,E15.8, 9H   VOLUME ,E15.8
   1 //,22H   LOWER FLUID DENSITY ,E15.8 ,25H DENSITY OF ULLAGE FLUID
   2 ,E15.8)
   ZCG=ZCG/RLGTH
   VOL=VOL/RLGTH**3
   MM1=M-1
   NM1=N-1
   DO 4110 J=2,NM1
   XTEMP=X(1,J+1)-X(1,J)
   IF(XTEMP.EQ.0.) GO TO 4108
   DZDR=(Y(1,J+1)-Y(1,J))/XTEMP
   IF(DZDR)4104,4102,4104
4102 SIGN=1.0
   GO TO 4106
4104 SIGN=DZDR/ABS(DZDR)
4106 DZDNF=SIGN/SQRT(1.0+DZDR**2)
   DRDNF=-DZDR*DZDNF
   GO TO 4109
4108 DZDNF=0.0
   DRDNF=-1.0
4109 VNF(J)=Y(1,J)*DRDNF-X(1,J)*DZDNF
4110 CONTINUE
   VNF(1)=0.0
   VNF(N)=Y(1,N)*DRDNF-X(1,N)*DZDNF
   MP1=M+1
   MP2=M+2
   DO 4120 I=2,M
   XTEMP=X(MP2-I,N)-X(MP1-I,N)
   IF(XTEMP.EQ.0.) GO TO 4118
   DZDR=(Y(MP2-I,N)-Y(MP1-I,N))/XTEMP
   IF(DZDR)4114,4112,4114
4112 SIGN=-1.0
   GO TO 4116
4114 SIGN=DZDR/ABS(DZDR)
4116 DZDNW=-SIGN/SQRT(1.0+DZDR**2)
   DRDNW=-DZDR*DZDNW
   GO TO 4119
4118 DZDNW=0.0
   DRDNW=1.0
4119 VNW(MP2-I)=Y(MP2-I,N)*DRDNW-X(MP2-I,N)*DZDNW
4120 CONTINUE
   VNW(1)=Y(1,N)*DRDNW-X(1,N)*DZDNW
   DO 4130 J=1,NM1
   XTEMP=X(M,J+1)-X(M,J)
   IF(XTEMP.EQ.0.) GO TO 4128
   DZDR=(Y(M,J+1)-Y(M,J))/XTEMP
   IF(DZDR)4124,4122,4124
4122 SIGN=1.0

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      GO TO 4126
4124 SIGN=DZDR/ABS(DZDR)
4126 DZDNW=-SIGN*DZDR/SQRT(1.0+DZDR**2)
      DRDNW=-DZDR*DZDNW
      GO TO 4129
4128 DZDNW=0.0
      DRDNW=1.0
4129 VNB(J)=Y(M,J)*DRDNW-X(M,J)*DZDNW
4130 CONTINUE
      VNB(N)=Y(M,N)*DRDNW-X(M,N)*DZDNW
      C(1,1,7)=0.0
      SS(3)=SQRT((X(1,N)-X(1,N-1))**2+(Y(1,N)-Y(1,N-1))**2)
      SS(4)=SQRT((X(2,N)-X(1,N))**2+(Y(2,N)-Y(1,N))**2)
      C(1,N,7)=.5*(VNF(N)+VNF(N-1))* .5*SS(3)*X(1,N)+
1      .5*(VNW(1)+VNW(2))* .5*SS(4)*X(1,N)
      DO 4132 J=2,NM1
      SS(3)=SQRT((X(1,J)-X(1,J-1))**2+(Y(1,J)-Y(1,J-1))**2)
      SS(6)=SQRT((X(1,J+1)-X(1,J))**2+(Y(1,J+1)-Y(1,J))**2)
      C(1,J,7)=VNF(J)*.5*(SS(3)+SS(6))*X(1,J)
4132 CONTINUE
      DO 4134 I=2,MM1
      SS(1)=SQRT((X(I,N)-X(I-1,N))**2+(Y(I,N)-Y(I-1,N))**2)
      SS(4)=SQRT((X(I+1,N)-X(I,N))**2+(Y(I+1,N)-Y(I,N))**2)
      C(I,N,7)=VNW(I)*.5*(SS(1)+SS(4))*X(I,N)
4134 CONTINUE
      SS(1)=SQRT((X(M,N)-X(M-1,N))**2+(Y(M,N)-Y(M-1,N))**2)
      SS(3)=SQRT((X(M,N)-X(M,N-1))**2+(Y(M,N)-Y(M,N-1))**2)
      C(M,N,7)=(.5*(VNW(M)+VNW(M-1))* .5*SS(3)
1      +.5*(VNB(N)+VNB(N-1))* .5*SS(1))*X(M,N)
      DO 4136 J=2,NM1
      SS(3)=SQRT((X(M,J)-X(M,J-1))**2+(Y(M,J)-Y(M,J-1))**2)
      SS(6)=SQRT((X(M,J+1)-X(M,J))**2+(Y(M,J+1)-Y(M,J))**2)
      C(M,J,7)=VNB(J)*.5*(SS(3)+SS(6))*X(M,J)
4136 CONTINUE
      SS(6)=SQRT((X(M,2)-X(M,1))**2+(Y(M,2)-Y(M,1))**2)
      C(M,1,7)=VNB(1)*.5*SS(6)*.25*(3.*X(M,1)+X(M,2))
      DO 4138 I=1,M
      DO 4138 J=1,N
      PHI(I,J)=0.0
4138 CONTINUE
      CALL SOREL(1)
      XTEMP=0.0
      DO 4140 J=1,NM1
      XTEMP=XTEMP+.5*(PHI(1,J)*VNF(J)*X(1,J)
1      +PHI(1,J+1)*VNF(J+1)*X(1,J+1))*
2      SQRT((Y(1,J+1)-Y(1,J))**2+(X(1,J+1)-X(1,J))**2)
4140 CONTINUE
      DO 4142 I=1,MM1
      XTEMP=XTEMP+.5*(PHI(I,N)*VNW(I)*X(I,N)
1      +PHI(I+1,N)*VNW(I+1)*X(I+1,N))*
2      SQRT((Y(I+1,N)-Y(I,N))**2+(X(I+1,N)-X(I,N))**2)
4142 CONTINUE
      DO 4144 J=1,NM1
      XTEMP=XTEMP+.5*(PHI(M,J)*VNB(J)*X(M,J)
1      +PHI(M,J+1)*VNB(J+1)*X(M,J+1))*
2      SQRT((Y(M,J+1)-Y(M,J))**2+(X(M,J+1)-X(M,J))**2)
4144 CONTINUE
      SIF=XTEMP*PI/VOL
      DO 415 J=1,NM1
      DRS(J)=X(1,J+1)-X(1,J)
415 CONTINUE
      DRS(N)=DRS(NM1)
      DFDR(1)=0.0
      DFRR(1)=2.0*(Y(1,2)-Y(1,1))/DRS(1)**2
      DO 417 J=2,NM1
      R(1)=X(1,J-1)

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R(2)=X(1,J)
R(3)=X(1,J+1)
YT(1)=Y(1,J-1)
YT(2)=Y(1,J)
YT(3)=Y(1,J+1)
CALL SFIT(FR,YTEMP,ZTEMP,XTEMP)
DFDR(J)=XTEMP
DFRR(J)=YTEMP
XK(J)=ZTEMP
417 CONTINUE
DFDR(N)=FR
DFRR(N)=DFRR(N-1)
XK(1)=XK(2)
XK(N)=YTEMP
DO 418 J=1,NM1
418 DS(J)=2.0*PI*(X(1,J+1)+X(1,J))*0.5
1 *SQRT((Y(1,J+1)-Y(1,J))**2+(X(1,J+1)-X(1,J))**2)
DS(N)=DS(NM1)
DO 419 J=1,NM1
DRW(J)=X(M,J+1)-X(M,J)
RW(J)=X(M,J)
ZW(J)=Y(M,J)
419 DZW(J)=Y(M,J+1)-Y(M,J)
RW(N)=X(M,N)
ZW(N)=Y(M,N)
DO 420 K=1,MM1
ITEMP=M-K+1
DRW(N+K-1)=X(M-K,N)-X(ITEMP,N)
DZW(N+K-1)=Y(M-K,N)-Y(ITEMP,N)
RW(N+K)=X(M-K,N)
420 ZW(N+K)=Y(M-K,N)
RW(N+M)=X(1,N)
ZW(N+M)=Y(1,N)
DRW(NPMM1)=DRW(N+M-2)
DZW(NPMM1)=DZW(N+M-2)
DRW(N+M)=DRW(NPMM1)
DZW(N+M)=DZW(NPMM1)
DO 422 J=1,NM2
XTEMP=0.0
DO 421 K=1,NM1
421 XTEMP=XTEMP+0.5*(PHIS(K,J)**2+PHIS(K+1,J)**2)*DS(K)
422 ALPJS(J)=XTEMP
DO 423 J=1,NM2
DO 423 I=1,NM2
ENU(I,J)=2.0*PI/ALPJS(I)*X(1,N)*PHIS(N,J)*PHIS(N,I)*EIGVAL(J)
423 CONTINUE
C READ NUMBER OF EIGENVALUES TO BE USED IN REMAINING CALCULATIONS
READ 95,NEV
NM2=NEV
DO 430 J=1,NM2
DO 430 I=1,NM2
EPSIJ=0.0
BETAIJ=0.0
DELIJ(I,J)=0.0
GAMIJ=0.0
XTEMP=0.0
YTEMP=0.0
IF(I.EQ.J) DELIJ(I,J)=1.0
DO 428 K=1,NM1
YPRP=(Y(1,K+1)-Y(1,K))/DRS(K)
EE=DRS(K)/ABS(DRS(K))
CURVA=1.0+YPRP**2
CURVB=SQRT(CURVA)
EPSIJ=EPSIJ+0.5*(PHIS(K,J)*PHIS(K,I)/((X(1,K)+.00001)**2)
1 *PHIS(K+1,J)*PHIS(K+1,I)/(X(1,K+1)**2))*DS(K)
DPSIIA=(PHIS(K+1,I)-PHIS(K,I))/DRS(K)

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      DPSIJA=(PHIS(K+1,J)-PHIS(K,J))/DRS(K)
      ZTEMP=YPRP**2/(CURVA*(0.5*(X(1,K)+X(1,K+1)))**2)+
1      0.5*(XK(K)**2+XK(K+1)**2)
      XTEMP=XTEMP+DPSIIA*DPSIJA*DS(K)/CURVA
      YTEMP=YTEMP-.5*ZTEMP*(PHIS(K,I)*PHIS(K,J)+PHIS(K+1,I)*PHIS(K+1,J))
1      *DS(K)
      BETAIJ=BETAIJ+0.5*(PHIS(K,I)*PHIS(K,J)+PHIS(K+1,I)*PHIS(K+1,J))
1      *DS(K)/CURVB*EE
428 CONTINUE
      EPSIJ=EPSIJ*EIGVAL(J)/ALPJS(I)
      GAMIJ=(XTEMP+YTEMP)*EIGVAL(J)/ALPJS(I)
      BETAIJ=BETAIJ*EIGVAL(J)/ALPJS(I)
430 RMX(I,J)=-GAMMA*ENU(I,J)+GAMIJ+EM*EM*EPSIJ+BOND*BETAIJ
1      *(DEN-RHOU)/DEN
      CALL MATINV(DELIJ,IROW,ICOL,NM2,NDIM,1.0E-06)
      DO 432 I=1,NM2
      DO 432 J=1,NM2
      XTEMP=0.0
      DO 431 K=1,NM2
431 XTEMP=XTEMP+RMX(I,K)*DELIJ(K,J)
432 EVEC(I,J)=XTEMP
      DO 433 I=1,NM2
      DO 433 J=1,NM2
433 RMX(I,J)=EVEC(I,J)
      KM2=NM2
      CALL EIGEN(NM2,RMX,EVI,EVEC,50,2,0,1,NDIM)
C      PUT EIGENVALUES AND VECTORS IN INCREASING ORDER
      DO 442 I=1,NM2
      NVAL(I)=I
      IF (RMX(I,I)) 440,440,441
440 KM2=KM2-1
      COMS(I)=10.0E+60
      GO TO 442
441 COMS(I)=RMX(I,I)
442 CONTINUE
      DO 445 I=1,NM2
      K=I
      DO 444 J=K,NM2
      IF (COMS(I)-COMS(J)) 444,444,443
443 XTEMP=COMS(J)
      II=NVAL(J)
      COMS(J)=COMS(I)
      NVAL(J)=NVAL(I)
      COMS(I)=XTEMP
      NVAL(I)=II
444 CONTINUE
445 CONTINUE
      DO 447 J=1,NM2
      K=NVAL(J)
      DO 447 I=1,NM2
      F(I,J)=EVEC(I,K)
447 CONTINUE
      PRINT 4005
4005 FORMAT (39H0 EIGENVALUES AND EIGENVECTORS AT LOW G)
      DO 448 I=1,NM2
      PRINT 402,I,COMS(I)
      PRINT 202,(F(J,I),J=1,NM2)
448 CONTINUE
      XTEMP=DFDR(N)
      RII=X(1,N)
      FII=Y(1,N)
      NPMM2=NPMM1-1
      L=N+M
      RWMAX=RW(1)
      ZWMAX=ZW(1)
      DO 449 I=2,L

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      IF (RW(I)-RWMAX) 4491,449,449
449  RWMAX=RW(I)
      ZWMAX=ZW(I)
4491 CONTINUE
      RFMAX=X(1,1)
      ZFMAX=Y(1,1)
      DO 4493 I=2,N
      IF (X(1,I)-RFMAX) 4493,4492,4492
4492 RFMAX=X(1,I)
      ZFMAX=Y(1,I)
4493 CONTINUE
      DO 453 J=1,NM1
      XTEMP=0.0
      DO 451 K=1,NPMM2
      IF (ZW(K)-ZWMAX) 450,450,4501
450  SIGN=1.0
      GO TO 4502
4501 SIGN=-1.0
4502 IF (DRW(K)) 4504,4503,4504
4503 DZDR=1.0E+07
      GO TO 4505
4504 DZDR=DZW(K)/DRW(K)
4505 YTEMP=SQRT(1.0+DZDR**2)
      IF (ABS(DZDR)-1.0E+05) 4506,4507,4507
4506 DRDN=SIGN*DZDR/YTEMP
      GO TO 4508
4507 DRDN=1.0
4508 DZDN=-SIGN/YTEMP
      XT(1)=0.5*(RW(K)+RW(K+1))
      YT(1)=0.5*(ZW(K)+ZW(K+1))
      XT(2)=0.5*(PHIW(K,J)+PHIW(K+1,J))
      YT(2)=SQRT(DRW(K)**2+DZW(K)**2)
      XTEMP=XTEMP+PI*XT(1)*XT(2)*(YT(1)*DRDN-XT(1)*DZDN)*YT(2)
451 CONTINUE
C  XTEMP NOW CONTAINS MUJ1
      DO 452 K=1,NM1
      IF (DFDR(K)) 4511,4512,4511
4511 SIGN=DFDR(K)/ABS(DFDR(K))
      GO TO 4513
4512 SIGN=1.0
4513 IF (ABS(DFDR(K))-1.0E+05) 4514,4515,4515
4514 DRDNA=-SIGN*DFDR(K)/SQRT(1.0+DFDR(K)**2)
      GO TO 4516
4515 DRDNA=-1.0
4516 DZDNA=SIGN/SQRT(1.0+DFDR(K)**2)
      IF (DFDR(K+1)) 4521,4522,4521
4521 SIGN1=DFDR(K+1)/ABS(DFDR(K+1))
      GO TO 4523
4522 SIGN1=1.0
4523 IF (ABS(DFDR(K+1))-1.0E+05) 4527,4528,4528
4527 DRDNB=-SIGN1*DFDR(K+1)/SQRT(1.0+DFDR(K+1)**2)
      GO TO 4529
4528 DRDNB=-1.0
4529 DZDNB=SIGN1/SQRT(1.0+DFDR(K+1)**2)
      XT=SQRT(DRS(K)**2+(Y(1,K+1)-Y(1,K))**2)
      XTEMP=XTEMP+0.5*PI*(PHIS(K,J)*(Y(1,K)*DRDNA-X(1,K)*DZDNA)*X(1,K)
1  +PHIS(K+1,J)*(Y(1,K+1)*DRDNB-X(1,K+1)*DZDNB)*X(1,K+1))*XT
452 CONTINUE
      AMU(J)=XTEMP
453 CONTINUE
      IF (NEV .GT. KM2) NEV=KM2
      DO 461 K=1,NEV
      DO 461 I=1,NEV
      TALSK(K)=0.0
      DO 460 J=1,NM1
      XTEMP=0.0

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      YTEMP=0.0
      XT=0.0
      YT=0.0
      DO 459 L=1,NEV
        XTEMP=XTEMP-F(L,K)*(-EIGVAL(L)*PHIS(J,L)*SQRT(1.0+DFDR(J)**2))
        YTEMP=YTEMP+F(L,I)*PHIS(J,L)
        XT=XT-F(L,K)*(-EIGVAL(L)*PHIS(J+1,L)*SQRT(1.0+DFDR(J+1)**2))
459    YT=YT+F(L,I)*PHIS(J+1,L)
460    TALSK(K)=(.5*XTEMP*YTEMP*X(1,J)+0.5*XT*YT*X(1,J+1))*ABS(DRS(J))
      1  +TALSK(K)
461    RMX(I,K)=2.0*PI*TALSK(K)
      CALL MATINV(RMX,IROW,ICOL,NEV,NDIM,1.0E-06)
      DO 464 K=1,NEV
        XTEMP=0.0
        DO 463 J=1,NM1
          XTEMP=XTEMP+0.5*(-EIGVAL(K)*PHIS(J,K)*X(1,J)
      1  -EIGVAL(K)*PHIS(J+1,K)*X(1,J+1))*DS(J)
463    CONTINUE
464    GJ(K)=-XTEMP
      DO 466 K=1,NEV
        TALSK(K)=0.0
        DO 465 J=1,NEV
465    TALSK(K)=TALSK(K)+F(J,K)*GJ(J)
466    CONTINUE
      DO 4670 I=1,NEV
        XTEMP=0.0
        DO 467 J=1,NEV
467    XTEMP=XTEMP+RMX(I,J)*TALSK(J)
4670   EKS(I)=XTEMP
      EMS=1.0
      ZS=0.0
      DO 468 K=1,NEV
468    SIGK(K)=GJ(K)
      DO 471 K=1,NEV
        XTEMP=0.0
        XT=0.0
        YT=0.0
        DO 470 J=1,NEV
          XTEMP=XTEMP+F(J,K)*SIGK(J)
470    XT=XT+F(J,K)*AMU(J)
        TK(K)=0.5*XTEMP
        ELK(K)=EKS(K)*XT
        EMK(K)=EKS(K)*TK(K)/VOL
        EMS=EMS-EMK(K)
        ZK(K)=RLGTH/VOL*ELK(K)/EMK(K)
        ZS=ZS+EMK(K)*ZK(K)
471    CONTINUE
      ZS=1.0/EMS*(ZCG-ZS)
      XTEMP=0.0
      DO 473 K=1,NEV
473    XTEMP=XTEMP+EMK(K)*ZK(K)**2
      SI0=SIF-(EMS*ZS**2+XTEMP)/RLGTH**2
      EMF=DEN*VOL*RLGTH**3
      PRINT 475,SIF,SI0,EMF
475    FORMAT(25H0 SIF-MOMENT OF INERTIA =,E15.8/
      1      25H0 SI0-MOMENT OF INERTIA =,E15.8/
      2      22H0 EMF-MASS OF LIQUID =,E15.8)
      PRINT 500
500    FORMAT (7H0 MK/MF/)
      PRINT 202,(EMK(K),K=1,NEV)
      PRINT 501
501    FORMAT (4H0 ZK/)
      PRINT 202,(ZK(K),K=1,NEV)
      PRINT 502,EMS,ZS
502    FORMAT(5H0M0= ,E15.8,7H Z0= ,E15.8)
      ENDFILE KUN

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STOP
END

IV-17

C

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SUBROUTINE SFIT(YR,XK3,XK,YR2)
FIT QUADRATIC THROUGH THREE POINTS
COMMON C(40,24,8),PHI(40,24),WB,EPS,ITA,N,M,W0,RHO,R(6),YT(7)
IF(ABS((YT(3)-YT(2))/(R(3)-R(2))).GT.1.0) GO TO 30
IF(ABS((YT(2)-YT(1))/(R(2)-R(1))).GT.1.0) GO TO 30
XTEMP=(R(1)**2-R(3)**2)*(R(2)-R(3))-(R(2)**2-R(3)**2)*(R(1)-R(3))
YTEMP=(YT(1)-YT(3))*(R(2)-R(3))-(YT(2)-YT(3))*(R(1)-R(3))
AD=YTEMP/XTEMP
AB=(YT(2)-YT(3)-AD*(R(2)**2-R(3)**2))/(R(2)-R(3))
YR=2.0*AD*R(3)+AB
YR2=2.0*AD*R(2)+AB
YRR=2.0*AD
XK=YRR/(1.0+YR2**2)**1.5
H2=R(2)-R(3)
H1=R(1)-R(3)
F2=YT(2)
F1=YT(1)
F0=YT(3)
YRR=2.0*(H2*(F1-F0)-H1*(F2-F0))/(H1*H2*(H1-H2))
XK3=YRR/(1.0+YR**2)**1.5
RETURN
30 YTEMP=(YT(1)**2-YT(3)**2)*(YT(2)-YT(3))-(YT(2)**2-YT(3)**2)*
1 (YT(1)-YT(3))
XTEMP=(R(1)-R(3))*(YT(2)-YT(3))-(R(2)-R(3))*(YT(1)-YT(3))
AD=XTEMP/YTEMP
AB=(R(2)-R(3)-AD*(YT(2)**2-YT(3)**2))/(YT(2)-YT(3))
YR=1.0/(2.0*AD*YT(3)+AB)
YR2=1.0/(2.0*AD*YT(2)+AB)
YRR=-2.0*AD*YR2**3
XK=YRR/(1.0+YR2**2)**1.5
H2=YT(2)-YT(3)
H1=YT(1)-YT(3)
F2=R(2)
F1=R(1)
F0=R(3)
RYY=2.0*(H2*(F1-F0)-H1*(F2-F0))/(H1*H2*(H1-H2))
YRR3=-RYY*YR**3
XK3=YRR3/(1.0+YR**2)**1.5
RETURN
END
```

```

SUBROUTINE SOREL(LL)
COMMON C(40,24,8),PHI(40,24),WB,EPS,ITA,N,M,W0,RHO,R(6),YT(7)
C  SUCCESSIVE OVER-RELAXATION TO SOLVE FOR PHI
  IL=LL
  WX=WB
  DO 135 MOST=1,ITA
    KIT=MOST
    SRX=0.0
    SX=0.0
    DO 120 I=IL,M
      DO 120 J=1,N
        PHIT=C(I,J,7)
        DO 108 K=1,6
          CT=C(I,J,K)
          IF (CT .EQ. 0.0) GO TO 108
          IF (K-2) 100,101,102
100      II=I-1
          JJ=J
          GO TO 107
101      II=I-1
          JJ=J-1
          GO TO 107
102      IF (K-4) 103,104,105
103      II=I
          JJ=J-1
          GO TO 107
104      II=I+1
          JJ=J
          GO TO 107
105      IF (K-5) 109,106,109
106      II=I+1
          JJ=J+1
          GO TO 107
109      II=I
          JJ=J+1
107      PHIT=PHIT+CT*PHI(II,JJ)
108      CONTINUE
          PHIT=PHIT/C(I,J,8)
          RX=WX*(PHI(I,J)-PHIT)
          PHI(I,J)=PHI(I,J)-RX
          SRX=SRX+RX**2
          SX=SX+PHI(I,J)**2
120      CONTINUE
          IF (SX .EQ. 0.0) GO TO 140
          IF (KIT-1) 126,126,127
126      SPRX=SRX
127      EPSCX=SQRT(SRX/SX)
          IF (EPSCX .LE. EPS) GO TO 140
          ETAX=SQRT(SRX/SPRX)
          ELX=(WX+ETAX-1.0)/(WX*SQRT(ETAX))
          IF (ABS(ELX)-1.0) 129,129,128
128      WAX=WX
          GO TO 130
129      WAX=2.0/(1.0+SQRT(1.0-ELX**2))-W0
130      WX=RHO*WAX+(1.0-RHO)*WX
          SPRX=SRX
135      CONTINUE
          IF(IL .EQ. 1) PRINT 200,KIT,EPSCX
          IF(IL .EQ. 2) PRINT 204,KIT,EPSCX
200      FORMAT (*1SOLUTION DID NOT CONVERGE FOR INFLUENCE COEFFICIENTS*/,
1          * AFTER *,I5,* ITERATIONS  EPSCX= *,E15.7)
204      FORMAT(*1SOLUTION DID NOT CONVERGE FOR EIGENVECTOR ON SURFACE*/,
1          * AFTER *,I5,* ITERATIONS  EPSCX= *,E15.8)

```

```
      DO 136 I=1,M
      PRINT 202,(PHI(I,J),J=1,N)
202  FORMAT (8E15.7)
136  CONTINUE
      STOP
140  IF(IL .EQ. 1) PRINT 203,KIT
      IF(IL .EQ. 2) PRINT 205,KIT
203  FORMAT (* INFLUENCE COEF. SOL. CONVERGED AFTER ITERATION*,IS)
205  FORMAT(* EIGENVECTOR SOLUTION CONVERGED AFTER ITERATION*,IS)
      RETURN
      END
```



```

SUBROUTINE EIGEN(N,B,TI,T,MAXIT,NDEC,N1OPT,N2OPT,NDIM)
C SUBROUTINE FOR GENERATING THE EIGENVALUES AND EIGENVECTORS
C OF A REAL SYMMETRIC OR NON-SYMMETRIC MATRIX.
C THIS PROGRAM GENERATES THE EIGENVALUE MATRIX (REAL OR COMPLEX),
C AND AS OPTIONS, THE EIGENVECTOR MATRIX AND ITS INVERSE.
C THE CALL FOR THIS SUBROUTINE IS AS FOLLOWS,
C CALL EIGEN (N,B,TI,T,MAXIT,NDEC,N1OPT,N2OPT,NDIM)
C WHERE N IS THE ORDER OF THE MATRIX
C B IS THE MATRIX WHOSE EIGENVALUES ARE DESIRED
C TI IS THE INVERSE OF THE EIGENVECTOR MATRIX
C T IS THE EIGENVECTOR MATRIX
C MAXIT IS MAX NO. OF ITERATIONS FOR GENERATING EIGENVALUES
C NDEC IS THE NUMBER OF TIMES RESULT IS REFINED
C N1OPT IS 1 IF OPTION =1 IS DESIRED, OTHERWISE 0.
C OPTION =1 PROVIDES FOR GENERATING THE EIGENVECTOR MATRIX INVERSE
C N2OPT IS 1 IF OPTION =2 IS DESIRED, OTHERWISE 0.
C OPTION =2 PROVIDES FOR GENERATING THE EIGENVECTOR MATRIX
C NDIM IS DIMENSIONED NO. OF ROWS OF MATRIX (B)
C THE ORGINAL MATRIX B IS LOST DURING THE COMPUTATIONS AND IS
C REPLACED BY THE EIGENVALUE MATRIX.
C DIMENSION B(1),TI(1),T(1)
C INITIALIZE COUNTERS FOR NO. OF ITERATIONS AND YR,YS REDUCTIONS
IT=0
NTIMES=0
ANORM=0.0
DO 1100 I=1,N
DO 1100 J=1,N
IJ=(J-1)*NDIM+I
1100 ANORM=ANORM+B(IJ)**2
ANORM=SQRTF(ANORM)
DO 1101 I=1,N
DO 1101 J=1,N
IJ=(J-1)*NDIM+I
1101 B(IJ)=B(IJ)/ANORM
C FORM IDENTITY MATRIX IN TI LOCATION IF OPTION 1 IS DESIRED
1000 IF(N1OPT)1010,1010,1001
1001 DO 1003 I=1,N
II = (I-1)*NDIM+I
DO 1002 J=1,N
IJ = (J-1)*NDIM+I
1002 TI(IJ) = 0.
1003 TI(II) = 1.0
C FORM IDENTITY MATRIX IN T LOCATION IF OPTION 2 DESIRED
1010 IF(N2OPT)1020,1020,1011
1011 DO 1013 I=1,N
II = (I-1)*NDIM+I
DO 1012 J=1,N
IJ = (J-1)*NDIM+I
1012 T(IJ) = 0.
1013 T(II) = 1.0
1020 CONTINUE
YR=10.0E-7
YS=10.0E-7
96 NO=N-1
SUM=10.0E20
97 TAU=0.0
EN=0.0
DO 1 I=1,NO
JO=I+1
DO 1 J=JO,N
IJ = (J-1)*NDIM+I
JI = (I-1)*NDIM+J
1 TAU = TAU+B(IJ)**2+B(JI)**2

```

```

      DO 2 I=1,N
      II = (I-1)*NDIM+I
      TE = B(II)
2    EN=EN+TE**2
4    EN=EN+TAU
5    DELN=SUM-EN
6    IF (DELN)8,8,7
7    SUM=EN
      IT=IT+1
      IF (MAXIT-IT)120,120,10
8    CONTINUE
      IF (NDEC-NTIMES)120,120,9
9    YR=YR/100.0
      YS=YS/100.0
      NTIMES=NTIMES+1
      IT=IT+1
10   DO 98 K=1,NO
      KK = (K-1)*NDIM+K
      KO=K+1
11   DO 98 M=KO,N
      MM = (M-1)*NDIM+M
      KM = (M-1)*NDIM+K
      MK = (K-1)*NDIM+M
      H=0.0
      G=0.0
      HJ=0.0
12   DO 24 I=1,N
13   IF (I-K)14,24,14
14   IF (I-M)15,24,15
15   IK = (K-1)*NDIM+I
      KI = (I-1)*NDIM+K
      IM = (M-1)*NDIM+I
      MI = (I-1)*NDIM+M
      BO = B(IK)
16   BR = B(KI)
17   BQ = B(IM)
18   BS = B(MI)
19   H=H+BR*BS-BO*BQ
20   TEP=BO*BO+BS*BS
21   TEM=BR*BR+BQ*BQ
22   G=G+TEP+TEM
23   HJ=HJ-TEP+TEM
24   CONTINUE
25   H=2.0*H
      D = B(KK)-B(MM)
      TEP = B(KM)
      TEM = B(MK)
      C=TEP+TEM
      E=TEP-TEM
26   IF (ABSF(C)-YR)27,27,30
27   CC=1.0
28   SS=0.0
29   GO TO 39
30   BY=D/C
31   IF (BY)400,401,401
400  SIG=-1.0
      GO TO 32
401  SIG =1.0
32   COT=BY+(SIG*SQRTF(BY*BY+1.0))
33   SS=SIG/SQRTF(COT*COT+1.0)
34   CC=SS*COT
35   TEP=CC*CC-SS*SS
36   TEM=2.0*SS*CC
37   D=D*TEP+C*TEM
38   H=H*TEP-HJ*TEM
39   CONTINUE

```

```

40 ED=2.0*E*D
41 EDH=ED-H
42 DEN=G+2.0*(E+E*D*D)
43 TEE=EDH/(DEN+DEN)
75 CONTINUE
205 IF (ABS(TEE)-YS)44,44,46
44 CH=1.0
45 SH=0.0
   GO TO 48
46 CH=1.0/SQRTF(1.0-TEE*TEE)
47 SH=TEE*CH
48 C1=CH*CC-SH*SS
49 C2=CH*CC+SH*SS
50 S1=CH*SS+SH*CC
51 S2=-CH*SS+SH*CC
52 CONTINUE
53 IF (S1)55,54,55
54 IF (S2)55,98,55
55 DO 59 J=1,N
   KJ = (J-1)*NDIM+K
   MJ = (J-1)*NDIM+M
56 B0 = B(KJ)
57 BR = B(MJ)
58 B(KJ) = C1*B0+S1*BR
59 B(MJ) = S2*B0+C2*BR
60 DO 66 J=1,N
   JK = (K-1)*NDIM+J
   JM = (M-1)*NDIM+J
61 B0 = B(JK)
62 BR = B(JM)
65 B(JK) = B0*C2-BR*S2
66 B(JM) = -B0*S1+BR*C1
1070 IF (N10PT)1075,1075,1071
1071 DO 1072 J=1,N
   KJ = (J-1)*NDIM+K
   MJ = (J-1)*NDIM+M
   BQ = TI(KJ)
   BS = TI(MJ)
   TI(KJ) = C1*BQ+S1*BS
1072 TI(MJ) = S2*BQ+C2*BS
1075 IF (N20PT)98,98,1076
1076 DO 1077 J=1,N
   JK = (K-1)*NDIM+J
   JM = (M-1)*NDIM+J
   B0 = T(JK)
   BR = T(JM)
   T(JK) = B0*C2-BR*S2
1077 T(JM) = -B0*S1+BR*C1
   98 CONTINUE
   GO TO 97
120 DO 1102 I=1,N
   DO 1102 J=1,N
   IJ=(J-1)*NDIM+I
1102 B(IJ)=B(IJ)*ANORM
   IF (N20PT) 1107,1107,1103
1103 DO 1106 J=1,N
   ANORM=0.0
   DO 1104 I=1,N
   IJ=(J-1)*NDIM+I
1104 ANORM=ANORM+T(IJ)**2
   ANORM=SQRTF(ANORM)
   DO 1105 I=1,N
   IJ=(J-1)*NDIM+I
1105 T(IJ)=T(IJ)/ANORM
1106 CONTINUE
1107 RETURN

```

```

SUBROUTINE MPRINT (A,M,N,MD)
DIMENSION A(1),IT(6),C(6)
EQUIVALENCE (IT,C)
2 FORMAT (1H0, 4X, 6( 6X, 7HCOLUMN 114 )   ///   )
3 FORMAT (1H 114, X, (6E 17.8) )
N1=N
N2=6
N3=6
N4=1
4 IF (N3-N1) 6,6,5
5 N2=N1-N3+6
N3=N1
6 K=0
DO 7 I= N4,N3
K=K+1
7 IT(K)=I
PRINT 2, (IT(I),I=1,N2)
DO 9 I=1,M
K=0
L=MD*(N4-1)+I
DO 8 J=N4,N3
K=K+1
C(K)=A(L)
L=L+MD
8 CONTINUE
9 PRINT 3, I, (C(K),K=1,N2)
IF (N3-N1) 10,11,11
10 N3=N3+6
N4=N4+6
GO TO 4
11 RETURN
END

```

SUBROUTINE MATINV(A,IROW,ICOL,N,NDIM,SMLST)	MATIN000
DIMENSION A(1),IROW(1),ICOL(1)	MATIN001
709-16065	MATIN002
709-16065 SUBROUTINE MATINV - MATRIX INVERSION ROUTINE	MATIN003
A = ARRAY NAME OF MATRIX	MATIN004
IROW = DIMENSIONED AT N+1 OR GREATER	MATIN005
ICOL = DIMENSIONED AT N OR GREATER	MATIN006
N = NUMBER OF EQUATIONS	MATIN007
NDIM = VALUE OF I IN DIMENSION A(I,J) , I AND J MAY DIFFER	MATIN008
SMLST = SMALLEST LEADING ELEMENT ALLOWED BEFORE CALLING THE	MATIN009
SYSTEM SINGULAR , USUALLY = 1.0 E-04 OR 1.0 E-05	MATIN010
	MATIN011
NP1=N+1	MATIN012
DO 5 I=1,N	MATIN013
ICOL(I)=I	MATIN014
5 IROW(I)=I	MATIN015
DO 75 ITER=1,N	MATIN016
MAXR=ITER	MATIN017
MAXC=1	MATIN018
TEMP=ABSF(A(MAXR))	MATIN019
LIMITC=NP1-ITER	MATIN020
DO 15 I=ITER,N	MATIN021
DO 15 J=1,LIMITC	MATIN022
IJ=(J-1)*NDIM+I	MATIN023
IF (TEMP-(ABSF(A(IJ)))) 10,15,15	MATIN024
10 MAXR=I	MATIN025
MAXC=J	MATIN026
TEMP=ABSF(A(IJ))	MATIN027
15 CONTINUE	MATIN028
IF (TEMP-SMLST) 20,20,25	MATIN029
20 IROW(NP1)=ITER	MATIN030
PRINT 200, ITER,SMLST	MATIN031
200 FORMAT (7H00N THEI3,63HTH ITERATION ALL THE REMAINING TERMS WERE L	MATIN032
1ESS THAN OR EQUAL TO E11,4,18H IN ABSOLUTE VALUE)	MATIN033
RETURN	MATIN034
25 IF (MAXR-ITER) 30,40,30	MATIN035
30 DO 35 J=1,N	MATIN036
MAXRJ=(J-1)*NDIM+MAXR	MATIN037
ITJ=(J-1)*NDIM+ITER	MATIN038
TEMP=A(MAXRJ)	MATIN039
A(MAXRJ)=A(ITJ)	MATIN040
35 A(ITJ)=TEMP	MATIN041
ITEMP=IROW(MAXR)	MATIN042
IROW(MAXR)=IROW(ITER)	MATIN043
IROW(ITER)=ITEMP	MATIN044
40 IF (MAXC-1) 45,55,45	MATIN045
45 DO 50 I=1,N	MATIN046
IMAXC=(MAXC-1)*NDIM+I	MATIN047
TEMP=A(I)	MATIN048
A(I)=A(IMAXC)	MATIN049
50 A(IMAXC)=TEMP	MATIN050
ITEMP=ICOL(MAXC)	MATIN051
ICOL(MAXC)=ICOL(1)	MATIN052
ICOL(1)=ITEMP	MATIN053
55 TEMP=A(ITER)	MATIN054
ITEMP=ICOL(1)	MATIN055
DO 60 J=2,N	MATIN056
ITJM1=(J-2)*NDIM+ITER	MATIN057
ITJ=(J-1)*NDIM+ITER	MATIN058
A(ITJM1)=A(ITJ)/TEMP	MATIN059
60 ICOL(J-1)=ICOL(J)	MATIN060
ITN=(N-1)*NDIM+ITER	MATIN061
	MATIN062

A(ITN)=1.0/TEMP	MATIN063
ICOL(N)=ITEMP	MATIN064
DO 75 I=1,N	MATIN065
IF (I-ITER) 65,75,65	MATIN066
65 TEMP=A(I)	MATIN067
DO 70 J=2,N	MATIN068
IJM1=(J-2)*NDIM+I	MATIN069
IJ=(J-1)*NDIM+I	MATIN070
ITJM1=(J-2)*NDIM+ITER	MATIN071
A(IJM1)=A(IJ)-A(ITJM1)*TEMP	MATIN072
70 CONTINUE	MATIN073
IN=(N-1)*NDIM+I	MATIN074
ITN=(N-1)*NDIM+ITER	MATIN075
A(IN)=- (TEMP*A(ITN))	MATIN076
75 CONTINUE	MATIN077
DO 100 I=1,N	MATIN078
DO 80 J=I,N	MATIN079
IF (IROW(J)-I) 80,85,80	MATIN080
80 CONTINUE	MATIN081
85 IF (I-J) 90,100,90	MATIN082
90 DO 95 L=1,N	MATIN083
LI=(I-1)*NDIM+L	MATIN084
LJ=(J-1)*NDIM+L	MATIN085
TEMP=A(LI)	MATIN086
A(LI)=A(LJ)	MATIN087
95 A(LJ)=TEMP	MATIN088
IROW(J)=IROW(I)	MATIN089
100 CONTINUE	MATIN090
DO 125 I=1,N	MATIN091
DO 105 J=I,N	MATIN092
IF (ICOL(J)-I) 105,110,105	MATIN093
105 CONTINUE	MATIN094
110 IF (I-J) 115,125,115	MATIN095
115 DO 120 L=1,N	MATIN096
IL=(L-1)*NDIM+I	MATIN097
JL=(L-1)*NDIM+J	MATIN098
TEMP=A(IL)	MATIN099
A(IL)=A(JL)	MATIN100
120 A(JL)=TEMP	MATIN101
ICOL(J)=ICOL(I)	MATIN102
125 CONTINUE	MATIN103
IROW(NP1)=0	MATIN104
RETURN	MATIN105
END	MATIN106

C INPUT DATA

TEST CASE 10C SPHERICAL TANK 3/4 FULL

BOND NUMBER = 10

12	12	200	0 1.8	1.3	.000001	1.0	10.
0.0	0.0	.1216	.0047	.2438	.0194	.3665	.0464
.4874	.0895	.6008	.1535	.6960	.2429	.7319	.2978
.7564	.3594	.7657	.4274	.7527	.5024	.7209	.5576
.768	.510	.825	.436	.884	.339	.933	.233
.970	.108	.995	-.037	.999	-.181	.977	-.341
.925	-.510	.855	-.647				
0.0	-1.131	.094	-1.126	.180	-1.113	.264	-1.093
.343	-1.070	.422	-1.036	.500	-0.997	.572	-0.949
.640	-0.898	.695	-0.845	.735	-0.805	.765	-0.774
0.0	-.056	0.0	-.116	0.0	-.181	0.0	-.273
0.0	-.369	0.0	-.484	0.0	-.581	0.0	-.702
0.0	-.821	0.0	-.976				
1.0	-65.788	5.0					
-.36708		3.14159267	1.0		0.0		

C INPUT DATA
 TEST CASE 11 SPHEROIDAL TANK 3/8 FULL ROND NUMBER = 5

9	12	200	n	1.8	1.3	.000001	1.0	5,	
0,0	0,0	.164	.008	.325	.032	.483	.075		
,633	,140	,768	.231	.827	.267	.877	.351		
,917	,422	.942	.500	.951	.581	.939	.663		
,964	,600	.980	.540	.9941	.454	1,00	.367		
,994	,280	.980	.194	.954	.111				
0,0	-.499	.151	-.489	.301	-.460	.447	-.407		
.588	-.333	.718	-.235	.750	-.201	.800	-.154		
,833	-.114	.865	-.070	.900	-.015	.923	.031		
0,0	-.06	0,0	-.12	0,0	-.18	0,0	-.24		
0,0	-.30	0,0	-.36	0,0	-.42				
1,0	-16,12	5,0							
=1394	1.35		1.0		0.0				

7

APPENDIX V.

LISTING AND INPUT SAMPLE OF SSHAPE

```

      PROGRAM SSHAPE (INPUT,OUTPUT,TAPE5=INPUT,TAPE6=OUTPUT)
C      PROJECT 02-1846-n2
C      A L G U S T   1 9 6 9
C      *LOW GRAVITY LIQUID-VAPOR INTERFACE SHAPES IN
C      AXISYMMETRIC CONTAINERS *
C      C E C   6 6 0 0   F O R T R A N
      DIMENSION Y(10),F(10)
      DIMENSION XOR(2000),ZOR(2000),          YBX(200),YBZ(200)
      DIMENSION TBOX(2),TBANG(2),ANS(3),ARG(3)
      DIMENSION XYB(200),XRB(200),XY(2000),XR(2000)
      DIMENSION XKF(3),ZKF(3),XKB(3),ZKB(3)
      DIMENSION S(2000),G(50),RS(50),SSV(50),NN(2),RX(50),Z(50)
      TYPE REAL  KN,KUSV,KUP,K2F,K2B
      CON1 = 1.74532925E-02
      CON2 = 57.2957795
      PI = 3.14159265
      NTAPE1 = 5
      NTAPE0 = 6
C*****R E A D   C O N T A I N E R   C O O R D I N A T E S
      2000 READ (NTAPE1,500) NP
      500 FORMAT ( 2I5 )
      READ (NTAPE1,200) (YBX(I),I=1,NP)
      2000 FORMAT ( 8F10,0 )
      READ (NTAPE1,200) (YBZ(I),I=1,NP)
C*****C O N T A I N E R   V O L U M E
      NP1 = NP-1
      VT = 0,
      DO 95 I=1,NP1
      VT = VT+.5*(YBX(I)**2+YBX(I+1)**2)*(YBZ(I)-YBZ(I+1))
      95 CONTINUE
      VT = PI*VT
      20 READ (NTAPE1,200) ALPHA,K0,Y0,BN,BETAD,DK0,DBC,DY0
      READ (NTAPE1,200) THETA,DTHETA,RLGTH
C*****N ' U M B E R   O F   E Q U A T I O N S
      READ (NTAPE1,500) N,IBOPT
      READ (NTAPE1,500) NN(1),NN(2)
      DO 94 I=1,NP
      YBX(I) = YBX(I)/RLGTH
      YBZ(I) = YBZ(I)/RLGTH
      94 CONTINUE
      Y0 = Y0/RLGTH
      DY0 = DY0/RLGTH
      TPRINT = 2,
      IPRINT = (TPRINT+DTHETA/10,)/DTHETA)
      KUSV = K0
      DK0SV = DK0
      DK0 = 4, *DK0
      YBX1 = YBX(1)
      YBZ1 = YBZ(1)
      II = 0
      JJ = 0
      JK = 0
      TBOX(2) = K0
      1000 NT = 0
      Y(1) = Y0
      Y(2) = 0,
      X = THETA*CON1
      H = DTHETA*CON1
      WRITE (NTAPE0,300)
      300 FORMAT (22H1 I N P U T   D A T A)
      WRITE (NTAPE0,305) ALPHA,K0,Y0,BN
      305 FORMAT (9H0ALPHA = ,E15,8,6H (DEG),4X,5HK0 = ,E15,8,6X,5HY0 = ,

```

```

1 E15.8,4X,5HBN = ,E15.8) SSH06300
WRITE (NTAPE0,310) SSH06400
310 FORMAT (1H0,5X,5HTHETA,10X,3HXOR,12X,3HZOR,12X,4HY(1),12X,4HY(2)/) SSH06500
XOR(1) = 0. SSH06600
ZOR(1) = Y0 SSH06700
XDEG = 0, SSH06800
WRITE(NTAPE0,315) XDEG,XOR(1),ZOR(1),Y(1),Y(2) SSH06900
VU1 = 0, SSH07000
ICOUNT = 1 SSH07100
K = 1 SSH07200
IJK = 2 SSH07300
DO 35 I=2,NP SSH07400
75 F(1) = Y(2) SSH07500
IF ( X,EQ,0, ) 125,130 SSH07600
125 F(2) = Y0*(1,-KN) SSH07700
GO TO 135 SSH07800
130 CONTINUE SSH07900
TERM = Y(1)**2+Y(2)**2 SSH08000
IF ( TERM-1,0E+60 ) 131,132,132 SSH08100
132 XDEG = X*CON2 SSH08200
WRITE (NTAPE0,315) XDEG,Y(1),Y(2),F(1),F(2) SSH08300
IF ( JJ,EQ,0 ) 777,778 SSH08400
777 KN = K0-DK0SV SSH08500
GO TO 1000 SSH08600
778 CONTINUE SSH08700
GO TO 74 SSH08800
131 CONTINUE SSH08900
F(2) = ((2.*Y(1)**2+3.*Y(2)**2)/Y(1))-(Y(2)/Y(1)**2)*(1,/ SSH09000
1 TAN(X))*Y(1)**2+Y(2)**2)*(1,/Y(1))*(8N*(Y(1)*COS(X)-Y0)- SSH09100
2 ((2.*KN)/Y0))*(SQRT(Y(1)**2+Y(2)**2))**3 SSH09200
135 CONTINUE SSH09300
S = RKLDEQ ( N, Y, F, X, H, NT ) SSH09400
IF ( S-1,0 ) 105,75,110 SSH09500
105 STOP SSH09600
110 TERM1 = SIN(X) SSH09700
TERM2 = COS(X) SSH09800
XDEG = X*CON2 SSH09900
TERM = YBZ(I-1)-YBZ(I) SSH10000
IF ( TERM,EQ,0, ) 465,470 SSH10100
465 R = 1,0E+100 SSH10200
GO TO 475 SSH10300
470 R = (YBX(I)-YBX(I-1))/TERM SSH10400
475 CONTINUE SSH10500
YX = (YBX(I-1)+R*YBZ(I-1))/(TERM1+R*TERM2) SSH10600
YBX2 = YBX(I) SSH10700
YBZ2 = YBZ(I) SSH10800
YZ = SQRT(YBX(I)**2+YBZ(I)**2) SSH10900
K = K+1 SSH11000
XOR(K) = Y(1)*TERM1 SSH11100
ZOR(K) = Y(1)*TERM2 SSH11200
VU1 = VU1+.5*(XOR(K-1)**2+XOR(K)**2)*(ZOR(K-1)-ZOR(K))*PI SSH11300
IF ( YZ,EQ,0, ) 111,113 SSH11400
111 THETAB = 90,*CON1 SSH11500
GO TO 112 SSH11600
113 THETAB = ASIN(YBX(I)/YZ) SSH11700
112 CONTINUE SSH11800
IF ( Y(1)=YX ) 40,45,45 SSH11900
40 CONTINUE SSH12000
IF ( XDEG=90, ) 50,74,74 SSH12100
50 CONTINUE SSH12200
IF ( ICOUNT=IPRINT ) 115,120,115 SSH12300
115 ICOUNT = ICOUNT+1 SSH12400

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```

GO TO 117
120 CONTINUE
WRITE (NTAPE0,315) XDEG,XOR(K),ZOR(K),Y(1),Y(2)
315 FORMAT ( 8E15,7 )
      ICCUNT = 1
117 IF ( X-THETAB ) 116,118,118
116 YBX1 = YBX2
      YBZ1 = YBZ2
      GO TO 75
118 ISAVE = I
      GO TO 35
45 GAMMA = (ATAN2(Y(2)*SIN(X)+Y(1)*COS(X),Y(1)*SIN(X)-Y(2)*COS(X)))*
1 CCN2
      IF ( ABS(GAMMA)-90, ) 46,46,74
46 CONTINUE
      ARG1 = -(YBX(I)-YBX(I-1))
      ARG2 = -(YBZ(I)-YBZ(I-1))
      IF ( ABS(ARG2),LE,(1,0E-08)) 47,48
47 PHI = 90,
      GO TO 49
48 PHI = ATAN2(ARG1,ARG2)*CON2
49 ANGLE = GAMMA+PHI
      WRITE (NTAPE0,315) XDEG,XOR(K),ZOR(K),Y(1),Y(2)
      WRITE (NTAPE0,320) GAMMA,PHI,ANGLE,Y0,K0
320 FORMAT (10H0 GAMMA = ,E15.8,8H PHI = ,E15.8,10H ALPHA = ,E15.8,
1 7H Y0 = ,E15.8,7H K0 = ,E15.8)
      IF ( ABS(ANGLE-ALPHA)-,5 ) 55,55,60
55 VU2 = 0,
      JJ = 0
      JK = 0
      TBOX(2) = KOSV
      YBXSV = YBX(ISAVE-1)
      YBZSV = YBZ(ISAVE-1)
      YBX(ISAVE-1) = XOR(K)
      YBZ(ISAVE-1) = ZOR(K)
      IJ = ISAVE-1
      DO 100 J=IJ,NP1
      VU2 = VU2+,5*(YBX(J)**2+YBX(J+1)**2)*(YBZ(J)-YBZ(J+1))*PI
100 CONTINUE
      YBX(ISAVE-1) = YBXSV
      YBZ(ISAVE-1) = YBZSV
      VU1 = VU1*RLGTH**3
      VU2 = VU2*RLGTH**3
      VU = VU1+VU2
      VL = VT*VU
      BETA = VU/VT
      WRITE (NTAPE0,325) VU1,VU2,VL,VL,VT,BETA
325 FORMAT (8H0 VU1 = ,E15.8,8H VU2 = ,E15.8,7H VU = ,E15.8,
1 7H VL = ,E15.8,7H VT = ,E15.8,9H BETA = ,E15.8)
      BB = BETAD-BETA
      IF ( IBOPT,EQ,0 ) GO TO 145
      IF ( ABS(BB)-DBC ) 145,145,140
140 CONTINUE
      IF ( II ) 155,150,155
150 II = II+1
      ANS(1) = Y0
      Y0 = Y0+DY0
      ARG(1) = BETA
      K0 = KOSV
      DK0 = DKOSV
      GO TO 1000
155 IF ( II-1 ) 165,160,165

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SSH12500
SSH12600
SSH12800
SSH12900
SSH13000
SSH13200
SSH13300
SSH13400
SSH13500
SSH13600
SSH13700
SSH13800
SSH13900
SSH14000
SSH14100
SSH14200
SSH14300
SSH14400
SSH14500
SSH14600
SSH14700
SSH14800
SSH14900
SSH15000
SSH15100
SSH15200
SSH15300
SSH15400
SSH15500
SSH15600
SSH15700
SSH15800
SSH15900
SSH16000
SSH16100
SSH16200
SSH16300
SSH16400
SSH16500
SSH16600
SSH16700
SSH16800
SSH16900
SSH17000
SSH17100
SSH17200
SSH17300
SSH17400
SSH17500
SSH17600
SSH17700
SSH17800
SSH17900
SSH18000
SSH18100
SSH18200
SSH18300
SSH18400
SSH18500
SSH18600
SSH18700
SSH18800

160	II = II+1	SSH18900
	ANS(2) = Y0	SSH19000
	ARG(2) = BETA	SSH19100
	Y0 = Y0+DY0	SSH19200
	K0 = K0SV	SSH19300
	DK0 = DK0SV	SSH19400
	GO TO 1000	SSH19500
165	IF (II=2) 175,170,175	SSH19600
170	ANS(3) = Y0	SSH19700
	II = II+1	SSH19800
175	ARG(3) = BETA	SSH19900
	IF (ARG(1)-ARG(3)) 405,400,400	SSH20000
400	AT = ARG(1)	SSH20100
	ARG(1) = ARG(3)	SSH20200
	ARG(3) = AT	SSH20300
	TS = ANS(1)	SSH20400
	ANS(1) = ANS(3)	SSH20500
	ANS(3) = TS	SSH20600
405	IF (ARG(1)-ARG(2)) 415,410,410	SSH20700
410	PP = ARG(1)	SSH20800
	ARG(1) = ARG(2)	SSH20900
	ARG(2) = PP	SSH21000
	TS = ANS(1)	SSH21100
	ANS(1) = ANS(2)	SSH21200
	ANS(2) = TS	SSH21300
415	IF (ARG(2)-ARG(3)) 425,420,420	SSH21400
420	ST = ARG(2)	SSH21500
	ARG(2) = ARG(3)	SSH21600
	ARG(3) = ST	SSH21700
	TS = ANS(2)	SSH21800
	ANS(2) = ANS(3)	SSH21900
	ANS(3) = TS	SSH22000
425	CONTINUE	SSH22100
	IF (BETAD-ARG(1)) 435,430,430	SSH22200
430	IF (BETAD-ARG(3)) 440,445,445	SSH22300
440	IF (BETAD-ARG(2)) 450,455,455	SSH22400
450	ANSE = (ANS(1)*(ARG(2)-BETAD)-ANS(2)*(ARG(1)-BETAD))/(ARG(2)-	SSH22500
	1 ARG(1))	SSH22600
	GO TO 460	SSH22700
455	ANSE = (ANS(2)*(ARG(3)-BETAD)-ANS(3)*(ARG(2)-BETAD))/(ARG(3)-	SSH22800
	1 ARG(2))	SSH22900
	GO TO 460	SSH23000
435	ANSE = (ANS(1)*(ARG(2)-BETAD)+ANS(2)*(BETAD-ARG(1)))/(ARG(2)-	SSH23100
	1 ARG(1))	SSH23200
	GO TO 460	SSH23300
445	ANSE = (ANS(2)*(ARG(3)-BETAD)+ANS(3)*(BETAD-ARG(2)))/(ARG(3)-	SSH23400
	1 ARG(2))	SSH23500
460	CONTINUE	SSH23600
	WRITE (NTAPE0,700) ANSE	SSH23700
700	FORMAT (20HN EXTRAPOLATED Y0 = ,E15,8)	SSH23800
	Y0 = ANSE	SSH23900
	ARG(1) = ARG(2)	SSH24000
	ARG(2) = ARG(3)	SSH24100
	ANS(1) = ANS(2)	SSH24200
	ANS(2) = ANS(3)	SSH24300
	ANS(3) = Y0	SSH24400
	DK0 = DK0SV	SSH24500
	K0 = K0SV	SSH24600
	GO TO 1000	SSH24700
145	CONTINUE	SSH24800
	S(1) = 0.	SSH24900
	SSV(1) = S(1)	SSH25000

```

NS = K-1
DO 1 L=2,NS
DELX = (XOR(L+1)-XOR(L))*RLGTH
DELY = (ZOR(L)-ZOR(L+1))*RLGTH
DELS = SQRT(DELX**2+DELY**2)
S(L) = S(L-1)+DELS
1 CONTINUE
DO 5 J=1,2
IF (NN(J),EQ,0) GO TO 6
NN1 = NN(J)-1
NM = NN(J)
SI = S(NS)/NN1
G(1) = 0,
RS(1) = 1,
DO 2 L=2,NN1
DO 3 J=1,NS
IF (S(J)-SI*(L-1)) 3,4,4
4 SSV(L) = S(J)
H1 = S(J+1)-S(J)
H2 = S(J+1)-S(J)
DEN = H1*H2*(H2-H1)
AP0 = -(H2**2-H1**2)/DEN
AP1 = H2**2/DEN
AP2 = -(H1**2)/DEN
APP0 = 2,/(H1*H2)
APP1 = (-2,*H2)/DEN
APP2 = (2,*H1)/DEN
RX(L) = XOR(J)*RLGTH
Z(L) = (Y0-ZOR(J))*RLGTH
RPS = (AP2*XOR(J+1)+AP0*XOR(J)+AP1*XOR(J-1))*RLGTH
RPSS = (APP2*XOR(J+1)+APP0*XOR(J)+APP1*XOR(J-1))*RLGTH
ZPS = (AP2*(Y0-ZOR(J+1))+AP0*(Y0-ZOR(J))+AP1*(Y0-ZOR(J-1)))*RLGTH
ZPSS = (APP2*(Y0-ZOR(J+1))+APP0*(Y0-ZOR(J))+APP1*(Y0-ZOR(J-1)))*
1 RLGTH
CKAP2 = ZPS/(XOR(J)*RLGTH)
CKAP1 = RPS*ZPSS-ZPS*RPSS
G(L) = CKAP2**2+CKAP1**2
RS(L) = RPS
GO TO 2
3 CONTINUE
2 CONTINUE
SSV(NM) = S(NS)
G(NM) = G(NN1)
RS(NM) = RS(NN1)
RX(NM) = XOR(K)*RLGTH
Z(NM) = (Y0-ZOR(K))*RLGTH
PRINT 314
314 FORMAT(1H1,14X,*G*,20X,*RS*,19X,*S*,19X,*XR*,19X,*XY*/)
WRITE (NTAPE0,319) (1,G(I),RS(I),SSV(I),RX(I),Z(I),I=1,NM)
319 FORMAT (15,5E20,8 )
5 CONTINUE
6 CONTINUE
YBX(1SAVE-1) = YBXSV
YBZ(1SAVE-1) = YBZSV
DO 605 J=1,NP
XRB(J) = YBX(J)*RLGTH
XYB(J) = (Y0-YBZ(J))*RLGTH
605 CONTINUE
XKF(1) = XOR(K-2)
XKF(2) = XOR(K-1)
XKF(3) = XOR(K)
ZKF(1) = -ZOR(K-2)+ZOR(1)

```

SSH25100
SSH25200
SSH25300
SSH25400
SSH25500
SSH25600
SSH25700
SSH25800
SSH25900
SSH26000
SSH26100
SSH26200
SSH26300
SSH26400
SSH26500
SSH26600
SSH26700
SSH26800
SSH26900
SSH27000
SSH27100
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SSH29700
SSH29800
SSH29900
SSH30000
SSH30100
SSH30200
SSH30300
SSH30400
SSH30500
SSH30600
SSH30700
SSH30800
SSH30900
SSH31000
SSH31100
SSH31200

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ZKF(2) = -ZOR(K-1)+ZOR(1)
ZKF(3) = -ZOR(K)+ZOR(1)
CALL SFIT (YR,XK3,XK,YR2,XKF,ZKF)
K2F = ABS(XK)
FRR2 = K2F*(1,+YR2**2)**(1,5)
XKB(1) = YBX(I-1)
XKB(2) = XOR(K)
XKB(3) = YBX(I)
ZKB(1) = -YBZ(I-1)+ZOR(1)
ZKB(2) = -ZOR(K)+ZOR(1)
ZKB(3) = -YBZ(I)+ZOR(1)
CALL SFIT (YR,XK3,XK,YR2,XKB,ZKB)
K2B = ABS(XK)
ZRR2 = K2B*(1,+YR2**2)**(1,5)
CGAM = (1,/SIN(ANGLE*CON1))*(K2B-COS(ANGLE*CON1)*K2F)
WRITE (NTAPEO,805) K2F,FRR2,K2B,ZRR2,CGAM
805 FORMAT (9H0 K2F = ,E15,8,9H FRR2 = ,E15,8//7H K2B = ,E15,8,
1 9H ZRR2 = ,E15,8//8H CGAM = ,E15,8)
DO 606 J=1,K
XR( J)= XOR(J)*RLGTH
XY( J)=(Y0-ZOR(J))*RLGTH
606 CONTINUE
Y0 = Y0*RLGTH
WRITE (NTAPEO,318) Y0
318 FORMAT (7H0 Y0 = ,E15,8)
WRITE (NTAPEO,335)
335 FORMAT (1H1,9X,3HXR0,17X,3HXY0/)
WRITE (NTAPEO,317) (XRB(J),XYB(J),J=1,NP)
317 FORMAT ( 2E20,8 )
SUM1 = 0,
DO 610 J=2,K
SUM1 = SUM1+,5*(XR(J)**2*XY(J)+XR(J-1)**2*XY(J-1))*(XY(J)-XY(J-1))
610 CONTINUE
SUM2 = 0,
IJ = ISAVE
XRBSV = XRB(IJ)
XYBSV = XYB(IJ)
XRB(IJ) = XOR(K)*RLGTH
XYB(IJ) = (Y0-ZOR(K))*RLGTH
DO 615 J=IJ,NP1
SUM2 = SUM2+,5*(XRB(J)**2*XYB(J)+XRB(J+1)**2*XYB(J+1))*(XYB(J+1)-
1 XYB(J))
615 CONTINUE
BZCGU = (PI*(SUM1+SUM2))/VT
XYB(IJ) = XYBSV
XRB(IJ) = XRBSV
SUM1 = 0,
DO 620 J=1,NP1
SUM1 = SUM1+,5*(XRB(J)**2*XYB(J)+XRB(J+1)**2*XYB(J+1))*(XYB(J+1)-
1 XYB(J))*PI
620 CONTINUE
ZCGT = SUM1/VT
ZCG = (ZCGT-BZCGU)/(1,-BETA)
WRITE (NTAPEO,800) ZCGT,BZCGU,ZCG
800 FORMAT (9H0 ZCGT = ,E15,8,10H BZCGU = ,E15,8,8H ZCG = ,E15,8)
GO TO 2000
60 CONTINUE
IF ( ANGLE-ALPHA ) 74,65,71
71 IF ( JJ#1 ) 72,73,72
72 TBOX(1) = K0
TBANG(1) = ANGLE
JJ = JJ+1

```

GO TO 70	SSH37500
73 TBCX(2) = KN	SSH37600
TBANG(2) = ANGLE	SSH37700
IF (JK) 76,76,77	SSH37800
77 KN = ,5*(KOP+TBDX(2))	SSH37900
GO TO 1000	SSH38000
76 KN = (TBDX(1)*(TBANG(2)-ALPHA)-TBDX(2)*(TBANG(1)-ALPHA))/	SSH38100
1 (TBANG(2)-TBANG(1))	SSH38200
JJ = 1	SSH38300
TBCX(1) = TBDX(2)	SSH38400
TBANG(1) = TBANG(2)	SSH38500
GO TO 1000	SSH38600
65 DK0 = DK0/4,	SSH38700
KN = K0 - DK0	SSH38800
GO TO 1000	SSH38900
70 DK0 = DK0/4,	SSH39000
KN = K0 + DK0	SSH39100
GO TO 1000	SSH39200
74 KOP = K0	SSH39300
KN = ,5*(K0+TBDX(2))	SSH39400
JK = 1	SSH39500
GO TO 1000	SSH39600
35 CONTINUE	SSH39700
END	SSH39800

FUNCTION RKLDEQ(N,Y,F,X,H,NT)	RKLDQ	1
C D2 UCSD RKLDEQ RUNGE-KUTTA-GILL LINEAR DIFFERENTIAL EQUATION SOLVER	F-63	X
C D2 UCSD RKLDEQ		F 63
C MODIFIED MAY 1963 (Q REMOVED FROM CALLING SEQUENCE)		
C TEST OF ALGOL ALGORITHM		
DIMENSION Y(10),F(10),G(10)	RKLDQ	2
C REAL X,H--INTEGER N,NT--COMMENT--BEGIN INTEGER I,J,L-REAL A	RKLDQ	3
NT=NT+1	RKLDQ	4
GO TO (1,2,3,4),NT		
C GO TO S(NT)		
1 DO 11 J=1,N	RKLDQ	5
11 Q(J)=0,	RKLDQ	6
A=,5	RKLDQ	7
X=X+H/2,	RKLDQ	8
GO TO 5	RKLDQ	9
2 A=,29289321881	RKLDQ	10
GO TO 5	RKLDQ	11
3 A=1,7071067812	RKLDQ	12
X=X+H/2,	RKLDQ	13
GO TO 5	RKLDQ	14
4 DO 41 I=1,N	RKLDQ	15
41 Y(I)=Y(I)+H*F(I)/6,-Q(I)/3,	RKLDQ	16
NT=0	RKLDQ	17
RKLDEQ=2,	RKLDQ	18
GO TO 6	RKLDQ	19
5 DO 51 L=1,N	RKLDQ	20
Y(L)=Y(L)+A*(H*F(L)-Q(L))	RKLDQ	21
51 Q(L)=2,*A*H*F(L)+(1,-3.*A)*Q(L)	RKLDQ	22
RKLDEQ=1,	RKLDQ	23
6 CONTINUE	RKLDQ	24
RETURN	RKLDQ	25
END	RKLDQ	26

```

SUBROUTINE SFIT(YR,XK3,XK,YR2,R,YT)
FIT QUADRATIC THROUGH THREE POINTS
DIMENSION R(3),YT(3)
IF(ABS((YT(3)-YT(2))/(R(3)-R(2))),GT.1,0) GO TO 30
IF(ABS((YT(2)-YT(1))/(R(2)-R(1))),GT.1,0) GO TO 30
XTEMP=(R(1)**2-R(3)**2)*(R(2)-R(3))-(R(2)**2-R(3)**2)*(R(1)-R(3))
YTEMP=(YT(1)-YT(3))*(R(2)-R(3))-(YT(2)-YT(3))*(R(1)-R(3))
AD=YTEMP/XTEMP
AB=(YT(2)-YT(3)-AD*(R(2)**2-R(3)**2))/(R(2)-R(3))
YR=2,0*AD*R(3)+AB
YR2=2,0*AD*R(2)+AB
YRR=2,0*AD
XK=YRR/(1,0+YR2**2)**1,5
H2=R(2)-R(3)
H1=R(1)-R(3)
F2=YT(2)
F1=YT(1)
F0=YT(3)
YRR=2,*(H2*(F1-F0)-H1*(F2-F0))/(H1+H2*(H1-H2))
XK3=YRR/(1,0+YR**2)**1,5
RETURN
30 YTEMP=(YT(1)**2-YT(3)**2)*(YT(2)-YT(3))-(YT(2)**2-YT(3)**2)*
1 (YT(1)-YT(3))
XTEMP=(R(1)-R(3))*(YT(2)-YT(3))-(R(2)-R(3))*(YT(1)-YT(3))
AD=XTEMP/YTEMP
AB=(R(2)-R(3)-AD*(YT(2)**2-YT(3)**2))/(YT(2)-YT(3))
YR=1,0/(2,0*AD*YT(3)+AB)
YR2=1,0/(2,0*AD*YT(2)+AB)
YRR=-2,0*AD*YR2**3
XK=YRR/(1,0+YR2**2)**1,5
H2=YT(2)-YT(3)
H1=YT(1)-YT(3)
F2=R(2)
F1=R(1)
F0=R(3)
RYY=2,*(H2*(F1-F0)-H1*(F2-F0))/(H1+H2*(H1-H2))
YRR3=-RYY*YR**3
XK3=YRR3/(1,0+YR**2)**1,5
RETURN
END

```

C I N P U T D A T A
C SPHEROIDAL TANK

37							
0,	0.0756	0.1509	0.2262	0.3010	0.3740	0.447	0.519
0.588	0.654	0.710	0.770	0.833	0.881	0.923	0.954
0.980	0.994	1.000	0.994	0.980	0.954	0.923	0.881
0.833	0.770	0.710	0.654	0.588	0.519	0.447	0.374
0.301	0.2262	0.1509	0.0756	0.			
1.732	1.730	1.722	1.710	1.693	1.668	1.640	1.607
1.566	1.520	1.468	1.411	1.347	1.277	1.202	1.122
1.039	0.953	0.866	0.779	0.693	0.610	0.530	0.455
0.385	0.321	0.264	0.212	0.166	0.125	0.092	0.064
0.039	0.022	0.010	0.002	0.			
5.	0.	0.50	5.	0.625	0.2	0.005	0.50
0.	0.05	1.0					
2	1						
17	33						